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Effect of Circular Arc Feet on a Control Law for a Biped

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1. Introduction

Over the past several years a considerable amount of studies have been proposed on biped walking. The choice of type of feet such as a contact points, flat feet and circular arc feet is important, because walking stability is essentially affected by the contact with the ground. Control methods of many traditional humanoids with flat foot are based on zero moment point (ZMP) that remains inside the convex hull of the foot support using the ankle torque. There are lots of successful results, but the gaits seem not to be so natural. On the other hand, for a biped with point contact a geometric tracking method for biped walking using input-output linearization (Aoustin & Formalsky, 1999; Grizzle et al., 2001; Aoustin & Formalsky, 2003; Chevallereau et al., 2003) produces stable gait that seems quite natural. (The idea of the geometric tracking can be seen in the previous studies of Furusho (Furusho et al., 1981) and Kajita (Kajita & Tani, 1991).) Grizzle, et al. (Grizzle et al., 2001) proposed the method for a three-link model, only two outputs are controlled, the reference are expressed as a function of the biped state. Zero dynamics with an impact event of the controlled system were analyzed by Poincaré method. The effectiveness of geometric tracking has been verified on a platform called 'Rabbit' (Chevallereau et al., 2003) (Fig.1 left) with point feet. Westervelt, et al. (Westervelt et al., 2005) gave some additional results to show capability for robustness, changing average walking rate, and rejecting a perturbation by 'one-step transition control' and 'event-based control'.

In the field of passive dynamic walking mechanisms (McGeer, 1990), it is shown that a biped with large radius circular arc feet can take easily a lot of steps. The prototype Emu (Fig.1 right) can be equipped with various arc feet with different radii (Kinugasa et al., 2003; 2007). In previous walking experiments the biped Emu is excited by gravity or forced oscillation of the length of legs. If the feet radius is 10% of leg length, the biped could only take few steps (Kinugasa et al., 2003) excited by the effect of gravity because of the sensitivity to disturbances produced by the cables, the guide to avoid lateral motion and so on. The biped could not walk by the forced oscillation. In the case of a radius which is 97% of leg length, the biped



Fig. 1. Biped bipeds, “Rabbit” (left) and “Emu” (right).

Emu (Fig.1 right) can take easily few dozen of steps (Kinugasa et al., 2007) by the gravity and the leg oscillations. The step number is limited only by the space of our laboratory. The effect of the radii of circular feet was significant for our results, but the change of radius is also accompanied by other difference in physical parameters, thus a direct conclusion on the experimental study is not obvious and a more rigorous study must be done. In fact, the same results are well known in the field of passive dynamic walking as it is mentioned in Section 2. The geometric tracking method that was used for the underactuated biped Rabbit can be extended to the case of underactuated biped with circular arc feet. If the biped has the circular arc feet, the analytical stability study given by Chevallereau, et al. (Chevallereau et al., 2003) can not be applied directly. The contact point between the supporting foot and the ground moves forward during the step in this case. The same difficulty appears also in a flat feet model. For this problem, Djoudi and Chevallereau (Chevallereau & Djoudi, 2006) gave a solution to analyze the stability with a chosen evolution of the ZMP.

The purpose of the paper is to show the effects of the circular arc feet for an underactuated planar biped controlled by a geometric tracking method. The effect of the feet shape on the control properties is obviously depending on the walking strategies. Therefore it is significant to clarify the effect of the feet shape on the geometric tracking even if it is well known in the passive dynamic walking field.

A model of our biped is composed of five links. Prismatic knee joints are employed to avoid the foot clearance problem which occurs in association with large foot, not actuated ankle and rotational knee joint. A geometric evolution of the biped configuration is controlled, instead of a temporal evolution. The input-output linearization with a PD control law and a feed forward compensation is used for geometric tracking. The temporal evolution is analyzed using Poincaré map. The map is given by an analytic expression based on the angular momentum about the mobile contact point. The effect of the radius of the circular arc feet on stability and the basin of attraction is revealed by analytic calculation. It is compared to the effect of radius of the circular arc feet on passive dynamic walking. Section 2 presents an overview of previous studies on the circular arc feet. Section 3 gives the biped model. It is composed of a dynamic model and the impact model (instantaneous double support). Section 4 presents the control method. Section 5 gives the stability analysis. Some simulation results are shown and some discussion on the effects of the feet radius is developed in Section 6. Section 7 concludes the paper.

2. Previous studies on biped with circular arc feet

A circular arc feet for the biped are often treated in the field of passive dynamic walking McGeer (1990). It is well known that a passive dynamic walking gives an extremely natural gait. McGeer showed that an eigenvalue of the “speed mode” came to unit when the radius of a circular arc foot approaches the length of legs, and the eigenvalue becomes unit for synthetic wheel which has the foot radius equals to the leg length. The “speed mode” was related to dissipation of energy at the impact.

Wisse, et al. Wisse & van Frankenhuyzen (2003) showed that the larger feet radius, the larger amount of disturbances is accepted in experiments. The robustness against disturbances is connected to the size of a basin of attraction for walking. Wisse explained in the other paper Wisse et al. (2005) that “The walker will fall backward if it has not enough velocity to overcome the vertical position. Circular feet smoothen the hip trajectory and thus relax the initial velocity requirement. As the result, the basin of attraction is enlarged.” However a decisive study on the effect of circular arc feet on the basin of attraction has yet to be performed. Recently, Wisse, et al. Wisse et al. (2006) presented a stability analysis of passive dynamic walking with flat feet and passive ankles. The effect of the flat feet was analogous to the effect of the circular arc feet for many properties in the sense that ZMP smoothly and monotonically moves forward from heel to toe. However he pointed out the need of validation for a more accurate model of the heel strike transition. Asano and Luo Asano & Luo (2007) discussed similar effect between the circular arc feet and the flat feet with actuated ankles.

Adamczyk, Collins and Kuo Adamczyk et al. (2006) studied the centre of mass (CoM) mechanical work per step with respect to foot radius for various simple models of biped powered by an instantaneous push-off impulse under the stance foot just before contralateral heel strike Kuo (2001). They also showed relationships between foot radius and metabolic costs from measured via respiratory gas exchange. The data are collected through human walking with feet attached to rigid arc, and they conclude that the most effective walking is obtained when the foot radius equals to 30% of leg length. Geometrically speaking, feet length should be at least twice of the product of the coxa angle between two legs and the radius of feet McGeer (1990). Therefore one might choose the radius as 1/3 of a leg length with an angle 0.3 rad between two legs, in order to make an anthropomorphic biped, as McGeer wrote.

Thus for anthropomorphic models, 1/3 of leg length seems to be desirable in the sense of geometry between step length and feet lengths McGeer (1990), “foot clearance problem” Wisse & van Frankenhuyzen (2003) and energy costs Adamczyk et al. (2006).

3. The biped modeling

A biped presented in Fig.2 is composed of a torso and two symmetric legs which consist of the prismatic frictionless knees and the circular arc feet. The hips are rotational frictionless joints. We assume that the contact point does not slip and the biped walks in a vertical sagittal plane. The vector $\theta = [l_1, l_2, \theta_1, \theta_2, \theta_3]'$ (“ ’ ” means transpose) of configuration variables (see Fig. 2, left) describes the shape of the biped during single support, l_i is the length of leg i , $\theta_i, i = 1, 2$ is the angle between the torso and the leg i , θ_3 is the absolute angle of the supporting leg. The contact point between the biped and the ground is N_1 . The lowest point of the swing leg tip is noted N_2 . The actuator torques and forces are expressed by a vector $\Gamma = [\Gamma_1, \Gamma_2, \Gamma_3, \Gamma_4]'$. The absolute orientation of the biped θ_3 is not directly actuated. Thus, in a single support (SS), the biped is an under-actuated system. The walking gait consists of single support phases

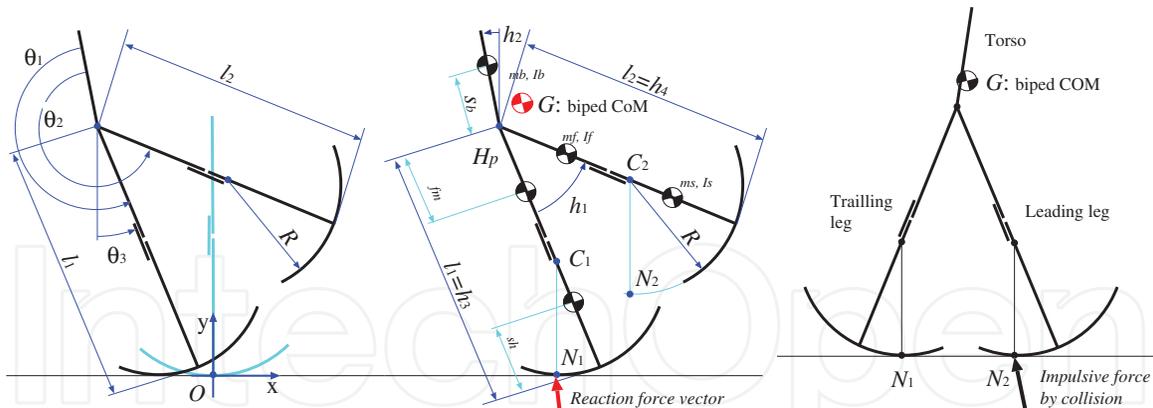


Fig. 2. The biped model: Left: coordinate of the model. Middle: physical parameters. Right: impact model.

separated by impacts, which are instantaneous double supports where a leg exchange takes place.

3.1 Dynamic model for single support phase

The dynamic model can be written as follows:

$$D(\theta)\ddot{\theta} + H(\theta, \dot{\theta}) = B\Gamma, \quad (1)$$

where $D \in \mathbb{R}^{5 \times 5}$ is the inertia matrix, the vector $H \in \mathbb{R}^5$ contains Coriolis, centrifugal and gravity terms. $B \in \mathbb{R}^{5 \times 4}$ defines how the inputs Γ enter the model. Due to the choice of joint coordinates, the matrix B is written as: $B = [I_4, O_{4 \times 1}]'$.

3.2 Impact model

To derive an impact model, an general dynamic model is written:

$$D_e(\theta)\ddot{\theta}_e + H_e(\theta_e, \dot{\theta}) = B_e\Gamma + D_{R_i}(\theta)R_i. \quad (2)$$

where $\theta_e = [\theta', x_H, y_H]'$, and x_H and y_H are the Cartesian coordinates of the hip position H_p shown in Fig.2 (right), $D_e \in \mathbb{R}^{7 \times 7}$ is the inertia matrix, the vector $H_e \in \mathbb{R}^7$ contains Coriolis, centrifugal and gravity terms. $R_i = [R_{x_i}, R_{y_i}]'$ is a ground reaction force vector applied at the contact point. $B_e \in \mathbb{R}^{7 \times 4}$ and $D_{R_i} \in \mathbb{R}^{7 \times 2}$ defines how the inputs Γ and R_i enter the model, i is the number of the leg in contact with the ground, $i = 1, i = 2$, or $i = 1, 2$.

When the leg i rolls on the ground, the contact with the ground occurs in N_i . If leg i touches the ground and since, we assume that no sliding occurs, the position of N_i is $ON_i = [-R\theta_3, 0]'$, where O is defined such that for the current step, the point contact is in 0 when θ_3 is zero. This position can also be calculated by : $ON_i = OH_p + H_pC_i + C_iN_i$ (Fig. 2, middle). Thus, we have :

$$\begin{bmatrix} -R\theta_3 \\ 0 \end{bmatrix} = \begin{bmatrix} x_H + (l_i - R) \sin \theta_3 \\ y_H - (l_i - R) \cos \theta_3 - R \end{bmatrix}. \quad (3)$$

Therefore, the following constraint equation is obtained:

$$\Psi_i := \begin{bmatrix} x_H + R\theta_3 + (l_i - R) \sin \theta_3 \\ y_H - R - (l_i - R) \cos \theta_3 \end{bmatrix} = 0. \quad (4)$$

Equation (4) is differentiated twice with respect to time, to obtain a constraint on the joint acceleration:

$$D'_{R_i} \ddot{\theta}_e + C_{R_i}(\theta_e, \dot{\theta}_e) \dot{\theta}_e = 0. \quad (5)$$

where $D'_{R_i} = \partial \Psi_i / \partial \theta_e$ and C_{R_i} comes from the derivation.

We assume that the impact is inelastic and instantaneous without sliding. Let $\dot{\theta}_e^-$ and $\dot{\theta}_e^+$ be the angular velocities just before and just after the impact, respectively. Let $I_{m_i} = [I_{mx_i}, I_{my_i}]'$, for $i = 1, 2$ be the vector of magnitudes of the impulsive reaction at the contact point of the stance and the swing leg. During the impact, the previous supporting leg can stay on the ground or take-off. If the leg takes off, the velocity of N_1 after the impact is positive. The impulsive ground reaction associated to a leg that stays on the ground must be positive and be in the friction cone. If the supporting leg takes off, the associated impulsive ground reaction is zero. The impact occurs when the leg tip of the swing leg contacts to the ground. To take into account the two cases, the following impact equation can be written:

$$\begin{cases} D_e(\theta)(\dot{\theta}_e^+ - \dot{\theta}_e^-) = D_R(\theta)I_m \\ D'_R(\theta)\dot{\theta}_e^+ = 0 \end{cases}, \quad (6)$$

where,

$$D_R(\theta) = \begin{cases} D_{R_2}(\theta), & \dot{y}_{N_1}^+ > 0 \\ D_{R_{12}}(\theta), & I_{my_1} > 0, I_{my_2} > 0 \end{cases}, \quad I_m = \begin{cases} I_{m_2}, & \dot{y}_{N_1}^+ > 0 \\ I_{m_{12}}, & I_{my_1} > 0, I_{my_2} > 0 \end{cases}$$

$$D_{R_{12}}(\theta) = \begin{bmatrix} D_{R_1}(\theta) & 0 \\ 0 & D_{R_2}(\theta) \end{bmatrix}, \quad I_{m_{12}} = \begin{bmatrix} I_{m_1} \\ I_{m_2} \end{bmatrix}.$$

From Eq. (6), we obtain:

$$\dot{\theta}_e^+ = (I_{7 \times 7} - D_e^{-1} D_R (D'_R D_e^{-1} D_R)^{-1} D'_R) \cdot \dot{\theta}_e^-. \quad (7)$$

Before and after the impact, the biped is in contact with the ground on at least one leg, thus x_H, y_H can be calculated as function of θ , and \dot{x}_H, \dot{y}_H can be calculated as function of $\dot{\theta}$. Equation (7) can be transformed into an equation of $\theta, \dot{\theta}$ only.

$$\dot{\theta}^+ = \Delta(\theta) \dot{\theta}^-, \quad (8)$$

where $\Delta(\theta) \in \mathbb{R}^{5 \times 5}$ is the impact matrix. This matrix depends on the foot radius R . In the gait studied, the legs swap their roles from one step to the next, thus since the biped is symmetric, the dynamic model is derived only for the support on leg 1. And the leg exchange is taken into account just after the impact. The state of the biped to begin the next step is :

$$\theta_i = T_{LS} \theta_f, \quad \dot{\theta}_i = T_{LS} \dot{\theta}^+, \quad \dot{\theta}^+ = \Delta(\theta_f) \dot{\theta}_f, \quad (9)$$

where $T_{LS} \in \mathbb{R}^{5 \times 5}$ is the permutation matrix describing the leg exchange, the indexes i, f denoted the initial and final states of the biped for one step.

4. Control law

Since the studied biped is underactuated, and since some significant results have been obtained for the control of underactuated biped with point contact Chevallereau et al. (2003); Westervelt et al. (2005), our strategy for walking is to control four variables, such that they track the reference defined with respect to the monotonic variable θ_3 . The four variables that are controlled are grouped in vector $h = [h_1, h_2, h_3, h_4]' = [\theta_2 - \theta_1, \theta_3 - \theta_1 + \pi, l_1, l_2]'$, composed of the angle between two legs, the absolute angle of the torso, and the leg lengths, (shown in Fig. 2, middle). This vector h , plus θ_3 defines the configuration of the biped. The relation with vector θ is the following:

$$\theta = \begin{bmatrix} h_3 \\ h_4 \\ -h_2 + \theta_3 \\ h_1 - h_2 + \theta_3 \\ \theta_3 \end{bmatrix} = \begin{bmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ 0 & -1 & 0 & 0 \\ 1 & -1 & 0 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix} h + \begin{bmatrix} 0 \\ 0 \\ 1 \\ 1 \\ 1 \end{bmatrix} \theta_3 \quad (10)$$

$$\theta = \frac{\partial \theta}{\partial h} h + \frac{\partial \theta}{\partial \theta_3} \theta_3. \quad (11)$$

where $\frac{\partial \theta}{\partial h}$ and $\frac{\partial \theta}{\partial \theta_3}$ are the constant matrices given in (10). Thus we have also:

$$\ddot{\theta} = \frac{\partial \theta}{\partial h} \ddot{h} + \frac{\partial \theta}{\partial \theta_3} \ddot{\theta}_3. \quad (12)$$

The control law is based on a computed torque control law and is such that the behavior of the controlled variables are:

$$\ddot{h} = \ddot{h}^d - K_p(h - h^d) - K_d(\dot{h} - \dot{h}^d). \quad (13)$$

But the reference to follow is a function of the variable θ_3 thus the reference is:

$$h^d = h^d(\theta_3) \quad (14)$$

$$\dot{h}^d = \frac{dh^d}{d\theta_3}(\theta_3)\dot{\theta}_3 \quad (15)$$

$$\ddot{h}^d = \frac{dh^d}{d\theta_3}(\theta_3)\ddot{\theta}_3 + \frac{d^2h^d}{d\theta_3^2}(\theta_3)\dot{\theta}_3^2, \quad (16)$$

Thus the desired behavior in closed loop is given by:

$$\ddot{h} = \frac{dh^d}{d\theta_3}(\theta_3)\ddot{\theta}_3 + \frac{d^2h^d}{d\theta_3^2}(\theta_3)\dot{\theta}_3^2 - K_p(h - h^d(\theta_3)) - K_d(\dot{h} - \frac{dh^d}{d\theta_3}(\theta_3)\dot{\theta}_3). \quad (17)$$

This expression is denoted:

$$\ddot{h} = \frac{dh^d}{d\theta_3}(\theta_3)\ddot{\theta}_3 + v(\theta, \dot{\theta}). \quad (18)$$

The dynamic model (1) can be expressed as function of \ddot{h} and $\ddot{\theta}_3$ using (12)

$$D(\theta)\left(\frac{\partial\theta}{\partial h}\ddot{h} + \frac{\partial\theta}{\partial\theta_3}\ddot{\theta}_3\right) + H(\theta, \dot{\theta}) = B\Gamma, \quad (19)$$

The torques will be calculated in order to have in closed loop the behavior given in (18), thus the torques must satisfy:

$$D(\theta)\left(\frac{\partial\theta}{\partial h}\frac{dh^d}{d\theta_3}(\theta_3) + \frac{\partial\theta}{\partial\theta_3}\ddot{\theta}_3 + \frac{\partial\theta}{\partial h}v(\theta, \dot{\theta})\right) + H(\theta, \dot{\theta}) = B\Gamma, \quad (20)$$

Since the biped is underactuated, all the motion are not possible and based on the expression of matrix B , the admissible acceleration $\ddot{\theta}_3$ can be deduced. The dynamic model is decomposed into two sub-models. The first sub-model is composed of the first four lines and allows to calculate the torque. The second sub-model is composed of the fifth line and allows to calculate $\ddot{\theta}_3$. This sub-system gives:

$$\ddot{\theta}_3 = \frac{-D_5(\theta)\frac{\partial\theta}{\partial h}v(\theta, \dot{\theta}) - H_5(\theta, \dot{\theta})}{D_5(\theta)\left(\frac{\partial\theta}{\partial h}\frac{dh^d}{d\theta_3}(\theta_3) + \frac{\partial\theta}{\partial\theta_3}\right)}, \quad (21)$$

where the index 5 refers to the 5th line of matrix D and vector H .

Finally, the control law is obtained:

$$\Gamma = D_{1,4}(\theta)\left(\frac{\partial\theta}{\partial h}\frac{dh^d}{d\theta_3}(\theta_3) + \frac{\partial\theta}{\partial\theta_3}\ddot{\theta}_3 + \frac{\partial\theta}{\partial h}v(\theta, \dot{\theta})\right) + H_{1,4}(\theta, \dot{\theta}), \quad (22)$$

where the indexes 1, 4 refer to the first four lines of matrix D and vector H .

5. Stability analysis

With the control, the output vector h converges to the reference path $h^d(\theta_3)$, and if the reference function is such that the impact condition is satisfied, the output is zero step after step for convenient choice of the control gains K_p, K_d Morris & Grizzle (2005).

5.1 Reference path

Since the initial and final configurations for a single support are double support configurations, when h^d is given, θ_3 can be deduced from geometrical relations. Thus the initial and final values of θ_3 on one step are known and denoted θ_{3i} and θ_{3f} . Since the condition of the impact is a geometrical condition, if the control law has converged and if θ_3 has a monotonic evolution, the configuration at the impact is the desired one. The reference function is designed such that the impact condition is satisfied. According to equations (8), (9), and (11), the reference path must be such that:

$$\theta(\theta_{3i}) = T_{LS}\theta(\theta_{3f}). \quad (23)$$

$$\left(\frac{\partial\theta}{\partial h}\frac{\partial h^d}{\partial\theta_3}(\theta_{3i}) + \frac{\partial\theta}{\partial\theta_3}\right)\dot{\theta}_{3i} = T_{LS}\Delta(\theta_{3f})\left(\frac{\partial\theta}{\partial h}\frac{\partial h^d}{\partial\theta_3}(\theta_{3f}) + \frac{\partial\theta}{\partial\theta_3}\right)\dot{\theta}_{3f}, \quad (24)$$

Equality (24) is composed of five scalar equations, thus $\frac{\partial h^d}{\partial\theta_3}(\theta_{3i})$ and $\frac{\dot{\theta}_{3i}}{\dot{\theta}_{3f}}$ can be calculated as function of $\frac{\partial h^d}{\partial\theta_3}(\theta_{3f})$. The ration of velocities is denoted $\delta_{\dot{\theta}_3}$:

$$\delta_{\dot{\theta}_3} = \frac{\dot{\theta}_{3i}}{\dot{\theta}_{3f}}. \quad (25)$$

5.2 Principle of the stability analysis

With the control law, the output vector h converges to the reference path $h^d(\theta_3)$. In the following section we assume that $h = h^d(\theta_3)$, that is, the system tracks the reference path. The five degrees of freedom (DoF) of the biped can be reduced to one DoF of a virtual equivalent pendulum under the condition, and we will hence analyze stability of the pendulum instead of the original biped.

This condition does not mean that the biped motion is cyclic with respect to time since the temporal evolution of θ_3 is the result of integration of Eq. (21), and thus depends on the reference path $h^d(\theta_3)$. For a SS phase θ_3 must evolve monotonically from θ_{3i} to θ_{3f} . The temporal evolution of the biped during a SS phase is completely defined by the velocity $\dot{\theta}_3$ for one particular value θ_3 . The stability analysis is based on the Poincaré return map, and this return map will be built just before the impact, when the biped is in the configuration $h^d(\theta_{3f}), \theta_{3f}$. The variable that is effective to study the convergence to a cyclic motion is $\dot{\theta}_{3f}$. Since the angular momentum is proportional to $\dot{\theta}_{3f}$, the angular momentum (or its square value) can also be used in the stability analysis

5.3 SS phase

According the Newton-Euler second law, as the gravity is the only external force that produces a torque around N_1 , the equilibrium of the biped in rotation around the mobile contact point N_1 gives:

$$\dot{\sigma}_{N_1} + MV_{N_1} \times V_G = r_{N_1G} \times M\vec{g}, \quad (26)$$

where V_{N_1} and V_G are the velocities at the points $N_1 = [-R\theta_3, 0]'$ and the center of mass, $G = [x_G, y_G]'$, M is the total mass of the biped, the gravity vector is $\vec{g} = [0, -g]'$, and σ_{N_1} is the angular momentum about N_1 . The general expression of σ_{N_1} is:

$$\sigma_{N_1} = \sum_i m_i r_{N_1G_i} \times V_{G_i} + \sum_i I_i w_i \quad (27)$$

where G_i is the center of mass for the link i , m_i and I_i are the mass and the inertia of link i , w_i is the angular velocity of link i , and V_{G_i} is the linear velocity of G_i . This quantity is linear with respect to the joint velocity component and can be written:

$$\sigma_{N_1} = S(\theta)\dot{\theta} \quad (28)$$

We assume that the biped follows reference path thus we have:

$$\theta = \frac{\partial \theta}{\partial h} h^d(\theta_3) + \frac{\partial \theta}{\partial \theta_3} \theta_3. \quad (29)$$

$$\dot{\theta} = \frac{\partial \theta}{\partial h} \frac{\partial h^d}{\partial \theta_3}(\theta_3) \dot{\theta}_3 + \frac{\partial \theta}{\partial \theta_3} \dot{\theta}_3. \quad (30)$$

Thus the angular momentum σ_{N_1} (28) is rewritten:

$$\sigma_{N_1} = S(\theta) \left(\frac{\partial \theta}{\partial h} \frac{\partial h^d}{\partial \theta_3}(\theta_3) + \frac{\partial \theta}{\partial \theta_3} \right) \dot{\theta}_3 = I_{\theta_3}(\theta_3) \dot{\theta}_3. \quad (31)$$

Equation (26) can be developed using the expression of r_{N_1G} , V_G , V_{N_1} as:

$$\dot{\sigma}_{N_1} = -Mg(x_G(\theta_3) + R\theta_3) + MR \frac{dy_G(\theta_3)}{d\theta_3} \dot{\theta}_3^2. \quad (32)$$

Equation (31) is combined to Eq. (32) to express the derivative of σ_{N_1} with respect to θ_3 , under the assumption that θ_3 is monotonic:

$$\frac{d\sigma_{N_1}}{d\theta_3} = -Mg(x_G + R\theta_3) \frac{I_{\theta_3}}{\sigma_{N_1}} + MR \frac{dy_G}{d\theta_3} \frac{\sigma_{N_1}}{I_{\theta_3}}. \quad (33)$$

A new variable $\xi = \sigma_{N_1}^2/2$ is introduced, to transform Eq. (33) into an equation that can be integrated analytically:

$$\begin{aligned} \frac{d\xi}{d\theta_3} &= \kappa_1(\theta_3) + 2\kappa_2(\theta_3)\xi, \\ \kappa_1(\theta_3) &= -Mg(x_G + R\theta_3)I_{\theta_3}, \\ \kappa_2(\theta_3) &= \frac{MR}{I_{\theta_3}} \left(\frac{\partial y_G(\theta)}{\partial \theta} \right)' \frac{d\theta^d}{d\theta_3}. \end{aligned} \quad (34)$$

Equation (34) is a first order ordinary differential equation linear in ξ . Therefore, a general solution can be obtained, for a step that begins with θ_{3i} as a initial value:

$$\xi(\theta_3) = \delta_{SS}^2(\theta_3)\xi(\theta_{3i}) + V(\theta_3), \quad (35)$$

$$\delta_{SS}(\theta_3) = \exp\left(\int_{\theta_{3i}}^{\theta_3} \kappa_2(\tau_2)d\tau_2\right), \quad (36)$$

$$V(\theta_3) = \int_{\theta_{3i}}^{\theta_3} \exp\left(\int_{\tau_1}^{\theta_3} 2\kappa_2(\tau_2)d\tau_2\right) \kappa_1(\tau_1)d\tau_1. \quad (37)$$

ξ and V are a pseudo-kinetic and a pseudo-potential energies of the virtual equivalent pendulum, respectively.

As a consequence if $\dot{\theta}_{3i}$ is known $\dot{\theta}_3$ can be deduced for the current step as a function of V and δ_{SS} without integration of (26). To be able to deduce from this equation the evolution of ξ (and in consequence of σ_{N_1} and θ_3) step after step, the evolution of ξ at the impact must be taken into account. In the following section, the index k will be added to denote the number of the current step

5.4 Impact phase

Let us consider the impact between steps k and $k + 1$. Using (31), ξ at the end of step k is:

$$\xi_k(\theta_{3f}) = \frac{1}{2}(I_{\theta_{3f}}(\theta_{3f})\dot{\theta}_{3f,k})^2 \quad (38)$$

and ξ at the beginning of the step $k + 1$ is:

$$\xi_{k+1}(\theta_{3i}) = \frac{1}{2}(I_{\theta_{3i}}(\theta_{3i})\dot{\theta}_{3i,k+1})^2 \quad (39)$$

Using (25), and defining δ_I by,

$$\delta_I = I_{\theta_3}(\theta_{3i})/I_{\theta_3}(\theta_{3f}), \quad (40)$$

we obtain:

$$\xi_{k+1}(\theta_{3i}) = \delta_I^2 \delta_{\theta_3}^2 \xi_k(\theta_{3f}). \quad (41)$$

5.5 Poincaré map

Combining (35) and (41), the final value of ζ from the k^{th} step to the $(k+1)^{\text{th}}$ step is as follows:

$$\tilde{\zeta}_{k+1}(\theta_{3f}) = \delta^2(\theta_{3f})\tilde{\zeta}_k(\theta_{3f}) + V(\theta_{3f}), \quad (42)$$

$$\delta(\theta_{3f}) = \delta_{SS}(\theta_{3f})\delta_I\delta_{\dot{\theta}_3}, \quad (43)$$

where θ_{3f} is the value of θ_3 just before the impact. This equation describes the Poincaré map. If a cyclic motion exists, then $\tilde{\zeta}_{k+1}(\theta_{3f})$ corresponds to $\tilde{\zeta}_k(\theta_{3f})$. Thus, a fixed point $\tilde{\zeta}_c(\theta_{3f})$ is given using (42) as follows:

$$\tilde{\zeta}_c(\theta_{3f}) = \frac{V(\theta_{3f})}{1 - \delta^2(\theta_{3f})}. \quad (44)$$

Since $\tilde{\zeta}_c(\theta_{3f})$ is positive, $V(\theta_{3f})$ and $1 - \delta^2(\theta_{3f})$ must have the same sign. The following cases can occur:

Case 1: From (42), the fixed point is stable, if $\delta^2(\theta_{3f}) < 1$. Therefore, if $\delta^2(\theta_{3f}) < 1$ and $V(\theta_{3f}) > 0$, then an asymptotically stable cyclic motion exists.

Case 2: If $\delta^2(\theta_{3f}) = 1$ and $V(\theta_{3f}) = 0$, from (42), $\tilde{\zeta}_{k+1}(\theta_{3f}) = \tilde{\zeta}_k(\theta_{3f})$, namely, all motions are cyclic.

Case 3: From (42), the fixed point is unstable, if $\delta^2(\theta_{3f}) > 1$. Therefore, if $\delta^2(\theta_{3f}) > 1$ and $V(\theta_{3f}) < 0$, then an unstable cyclic motion exists.

Case 4: $V(\theta_{3f})(1 - \delta^2(\theta_{3f})) < 0$, no cyclic motion exists.

Since by definition $\zeta \geq 0$, from Eq. (35) for the complete step, $\tilde{\zeta}_c$ must satisfy the following inequality:

$$\tilde{\zeta}_c(\theta_{3f}) \geq \tilde{\zeta}_{\min} = \max_{\theta_3} \frac{-V(\theta_3)}{\delta^2(\theta_3)}. \quad (45)$$

to have $\tilde{\zeta}(\theta_3) > 0$ for θ_3 between θ_{3i} and θ_{3f} .

Since a product of the two variables ($\delta_I \cdot \delta_{\dot{\theta}_3}$) is the ratio of momentum σ_{N_1} at the contact point N_1 before and after the impact, the speed of convergence is mainly associated with this ratio (This point will be detailed in the following sections), and connected to the distance between the contact points and velocity of the mass center before the impact Chevallereau et al. (2004). The contact point before the impact, at the end of the single support phase, is denoted N_1 , the contact point after the impact, at the beginning of the next single support phase, is denoted N_2 . Using equilibrium relation it is possible to compute the change of angular momentum around the contact point at impact as function of the value of the radii.

The distance d between the N_1 and N_2 is (see Fig.2)

$$N_1N_2 = d = 2(l - R) \sin(h_1/2). \quad (46)$$

The angular momentum before the impact denoted $\sigma_{N_1}^-$ is calculated around N_1 and can also be calculated around N_2 , it is then denoted $\sigma_{N_2}^-$, the angular momentum transfer gives:

$$\sigma_{N_2}^- = \sigma_{N_1}^- - M \cdot d \cdot \dot{y}_G^-. \quad (47)$$

m_s	1 [kg]	I_s	0.05[kgm ²]	s_h	0.4 [m]	l_1	0.8~0.85 [m]
m_f	1 [kg]	I_f	0.05[kgm ²]	f_m	0.2 [m]	l_2	0.75~0.8 [m]
m_b	15 [kg]	I_b	3[kgm ²]	s_b	0.1 [m]	R	0~1.0 [m]

Table 1. Physical parameters for the dynamic model

At the impact, considering the vertical component I_{my_1} of the impulsive ground reaction I_{m_1} in the point N_1 , the equilibrium in rotation around N_2 gives:

$$\sigma_{N_2}^+ = \sigma_{N_2}^- - d \cdot I_{my_1}, \quad (48)$$

where I_{my_1} is the vertical component of the impulsive ground reaction I_{m_1} applied by the ground in N_1 . The vertical equilibrium of the biped at the impact is :

$$I_{my_1} + I_{my_2} = M(\dot{y}_G^+ - \dot{y}_G^-), \quad (49)$$

where I_{my_1} and I_{my_2} are the vertical components of the impulsive ground reactions I_{m_1} and I_{m_2} respectively in the points N_1 and N_2 . The impact are such that the two legs stay on the ground, thus $I_{my_1} > 0$ and $I_{my_2} > 0$ and we have:

$$0 < I_{my_1} < M(\dot{y}_G^+ - \dot{y}_G^-). \quad (50)$$

As a consequence, combining (47), (48), and (50), we have:

$$\sigma_{N_1}^- - M \cdot d \cdot \dot{y}_G^+ < \sigma_{N_2}^+ < \sigma_{N_1}^- - M \cdot d \cdot \dot{y}_G^-, \text{ if } d > 0, \quad (51)$$

$$\sigma_{N_2}^+ = \sigma_{N_1}^-, \quad \text{if } d = 0, \quad (52)$$

$$\sigma_{N_1}^- - M \cdot d \cdot \dot{y}_G^- < \sigma_{N_2}^+ < \sigma_{N_1}^- - M \cdot d \cdot \dot{y}_G^+, \text{ if } d < 0. \quad (53)$$

When $I_{\theta_3} > 0$ (see Fig.7) and $\dot{\theta}_3 < 0$ (see Fig.4), $\sigma_{N_1}^- < 0$. Considering (25), (31) and (40), the ratio $\delta_I \delta_{\dot{\theta}_3}$ is bounded:

$$1 - M \cdot d \cdot \frac{\dot{y}_G^-}{\sigma_{N_1}^-} < \delta_I \delta_{\dot{\theta}_3} < 1 - M \cdot d \cdot \frac{\dot{y}_G^+}{\sigma_{N_1}^-}, \quad (d > 0), \quad (54)$$

$$\delta_I \delta_{\dot{\theta}_3} = 1, \quad (d = 0), \quad (55)$$

$$1 - M \cdot d \cdot \frac{\dot{y}_G^+}{\sigma_{N_1}^-} < \delta_I \delta_{\dot{\theta}_3} < 1 - M \cdot d \cdot \frac{\dot{y}_G^-}{\sigma_{N_1}^-}, \quad (d < 0) \quad (56)$$

6. Simulation

In simulations, the physical parameters of the biped shown in Fig.2 are used (see Table 1). The gains of the control law are chosen so that tracking errors can be smaller than 10^{-4} for all walking gaits.

$$\begin{cases} K_p = \text{diag}([10^5, 10^4, 10^5, 5 \times 10^4]) \\ K_d = \text{diag}([5 \times 10^2, 5 \times 10^2, 10^3, 5 \times 10^2]) \end{cases} \quad (57)$$

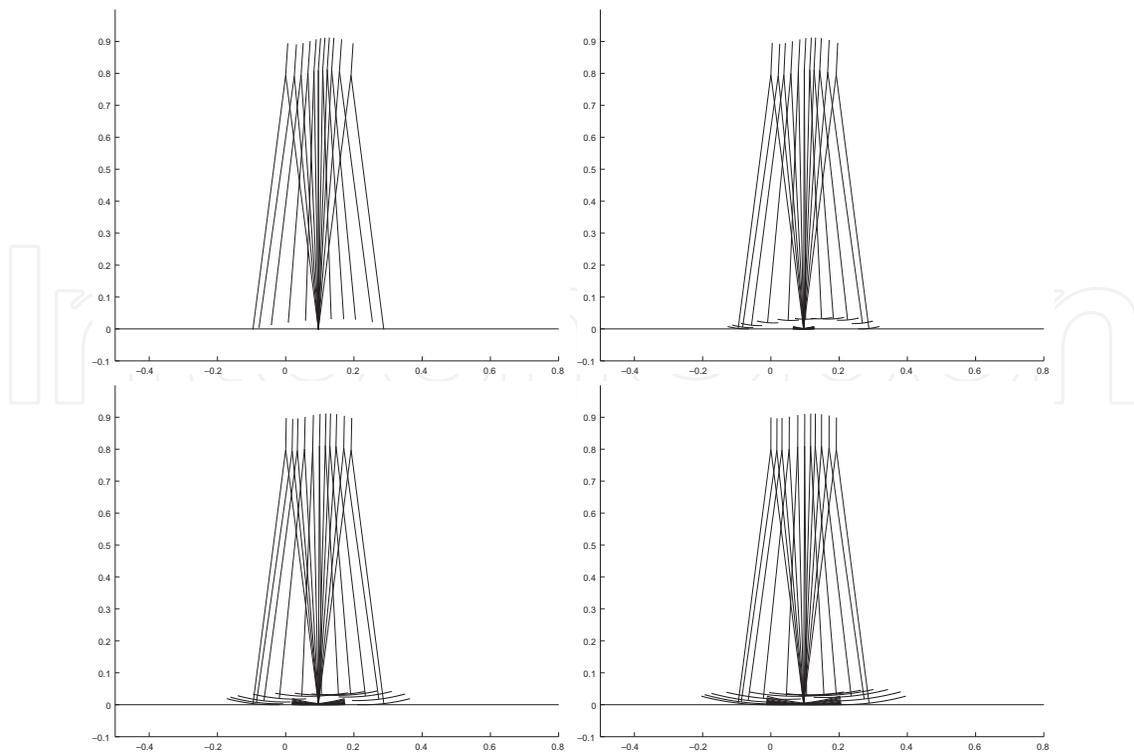


Fig. 3. The stick diagrams of walking. The foot radii $R = 0$ [m] (above left), 0.2 [m] (above right), 0.5 [m] (bellow left) and 0.7 [m] (bellow right).

Foot radius [m]	0	0.1	0.2	0.3	
Angle of torso [rad]	-0.060	-0.051	-0.043	-0.034	
Foot radius [m]	0.4	0.5	0.6	0.7	0.8
Angle of torso [rad]	-0.026	-0.018	-0.011	-0.004	0.002

Table 2. Torso angles. The angles are chosen such that cyclic motions have the same value $\zeta_c(\theta_{3f}) = \zeta(-0.12) = 16.27$.

6.1 Design of reference path

The reference path h^d is defined by a fourth order polynomial function such that:

$$h_d(\theta_3) = a[1, \theta_3^1, \theta_3^2, \theta_3^3, \theta_3^4]', \quad (58)$$

where $a \in \mathbb{R}^{4 \times 5}$ is a coefficient matrix for the reference h_d . An intermediate position of SS phase, positions and derivative with respect to θ just before the impact are given in order to calculate the coefficients of the reference paths (see Fig.3). Position and derivative with respect to θ after the impact are calculated by equations (23) and (24).

Walking is depending on not only the radii of feet but also of the reference path of the length of the legs. The foot radius reduces the velocity of the CoM before the impact. The reference paths of the legs are chosen to smoothen the vertical variation of the CoM. However the references of the legs are affected by the impact, and the choice of the reference paths is limited accordingly. The radius mainly smoothenes the vertical CoM motion.

The initial and the final length for the both legs are chosen as the same value. The final velocity for the biped are arbitrary fixed. The intermediate configuration for the legs is chosen such that the swing leg length decreases 0.02 m and the stance leg length increases 0.01 m during

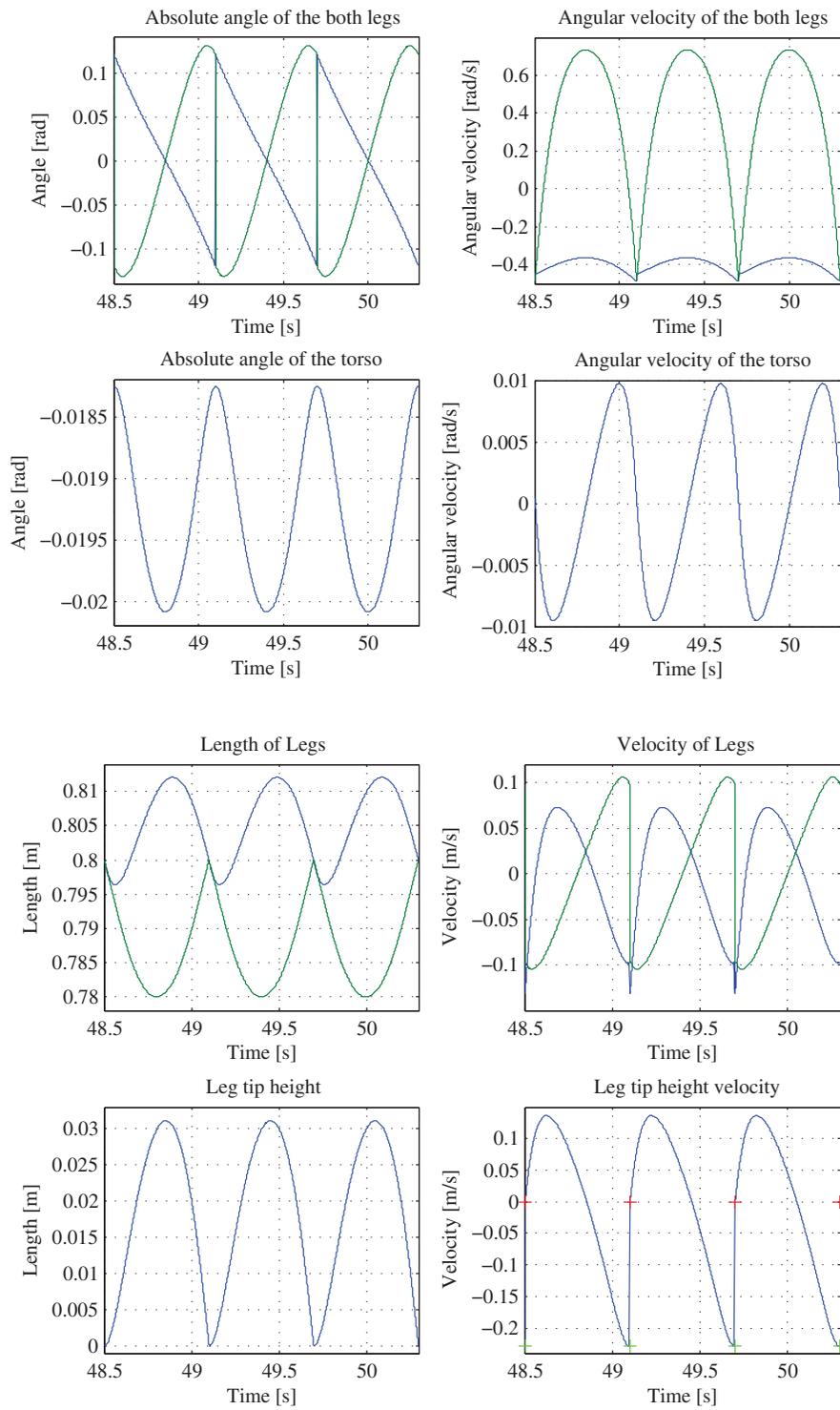


Fig. 4. Time responses at the cyclic motion with $R = 0.5$ [m] of the angle of the both legs, the torso, the length of legs and the leg tip. The reference paths are very well tracked.

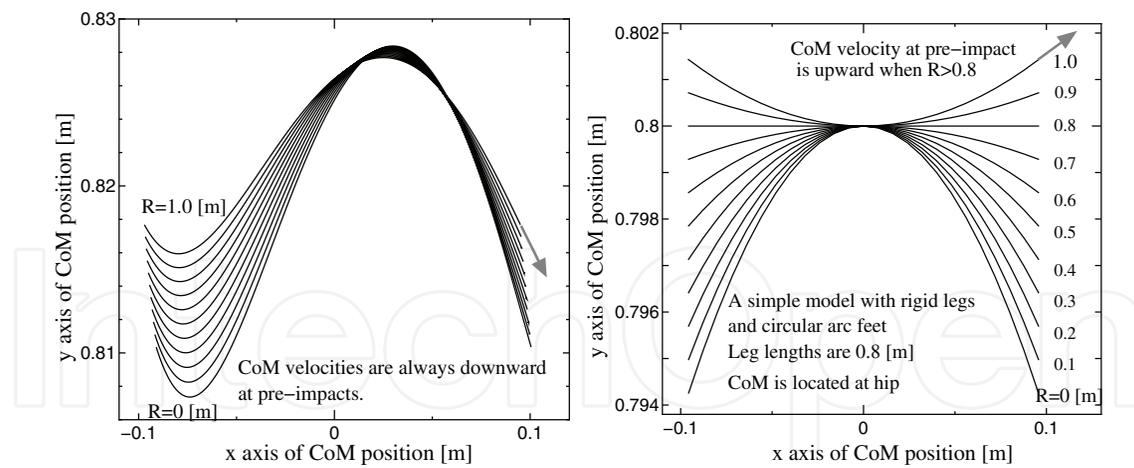


Fig. 5. CoM positions with respect to R . Left: the case of our biped shown in Fig.2. Tangent vectors of right ends of lines are expressing a post-impact velocity of CoM. Right: the case of a simple model with rigid legs and circular arc feet. CoM is located at hip position. When $R > 0.8$ [m], CoM velocities are upward. It gives a contradiction at the impact or there would be a flight phase.

the step to avoid that the swing leg tip touches the ground and the length of the legs is 0.8 [m] at the impact. Therefore the top position of the CoM is almost the same for each foot radius as shown in Fig.5. For one value R , we choose the angle of the torso at the impact arbitrary. The angle of the torso at the intermediate configuration is equal to 110% of the value of the torso angle at the impact. The corresponding value $\zeta_c(\theta_{3f})$ is deduced. For example, the coefficient matrix in Eq.(58) for $R = 0.5$ is obtained as follows:

$$a|_{R=0.5} = \begin{bmatrix} 0 & -3.02 & -0.158 & 70.8 & 10.9 \\ -0.0201 & 0.0002 & 0.255 & -0.0106 & -8.89 \\ 0.810 & -0.122 & -1.58 & 8.50 & 61.2 \\ 0.780 & -0.0037 & 1.91 & 0.254 & -36.5 \end{bmatrix} \quad (59)$$

Then from this reference motion we deduced the reference motion for the other value of the radius R . The angle of the torso at the impact $h_2(\theta_{3f})$ is adjusted such that the cyclic motions for all foot radii R have the same value $\zeta_c(\theta_{3f})$ as shown in Table 2.

Fig.3 shows examples of stick diagrams of walking for one step with the foot radii $R = 0$ [m], 0.2 [m], 0.5 [m] and 0.7 [m] and the step angle $= 0.24$ [rad]. A cyclic motion for $R = 0.5$ [m] is given in Fig.4. CoM positions with respect to R are shown in Fig.5. Tangent vectors of right ends of lines are expressing a post-impact velocity of CoM. The variation of CoM velocities at the impact are presented in Fig.6.

Energy excitation for continuous walking with smaller feet radius is mainly done by the asymmetric mass distribution due to the torso forward inclination. Leg swing also provides a way of putting energy. For small feet radii, the energy for walking is produced by the weight of the torso that is inclined forward. For larger feet radii, the energy for walking is produced by the motion of the swing leg.

Since the impact equation changes, the initial configuration and velocity are changed accordingly. During the impact, for the chosen reference path, the two legs stay on the ground.

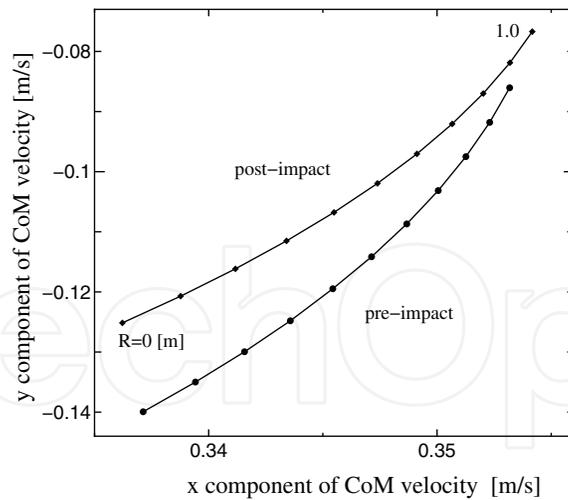


Fig. 6. CoM velocities at the impact with respect to R . The point corresponds to different value of R from 0 to 1, the abscissa of the point gives the horizontal velocity \dot{x}_G respectively before and after the impact, the ordinates gives the vertical velocity \dot{y}_G respectively before and after the impact. The vertical velocities before the impact are always directed downward.

6.2 Stability analysis

The variables in the analytic solution (35) are shown in Fig.7 with respect to the monotonic variable θ_3 for various values of the foot radius R . It should be noted that the monotonic variable is evolving from a positive value to a negative value, $\theta_3 : 0.12 \text{ [rad]} \rightarrow -0.12 \text{ [rad]}$. In Fig.7, $\zeta_c(\theta)$ is given for all the cyclic motions. It can be observed that $\zeta_c(\theta_{3f}) = \zeta(-0.12) = 16.27$. The figure of $\delta_{SS}^2(\theta_3)$ is given by Eq. (36). The convergence of Poincaré map, as shown in Eq. (43), is function of $\delta_{SS}^2(\theta_{3f}) = \delta_{SS}^2(-0.12)$. However the values of $\delta_{SS}^2(-0.12)$ are very close to unit thus the convergence of Poincaré map is essentially defined by the impact map : $\delta(\theta_{3f}) \approx \delta_I \delta_{\dot{\theta}_3}$. The second figure from the left of Fig.7 represents the evolution of V defined by Eq. (37). These functions are essentially affected by the evolution ζ . The third figure of Fig.7 shows the term I_{θ_3} given by Eq. (31), I_{θ_3} is always positive and has not large variation. This first study concerns reference path with an interlink angle at the impact equals to 0.24 [rad]. For this value, the evolution of $\delta_{SS}^2(\theta_{3f})$, δ_I , $\delta_{\dot{\theta}_3}$ and $\delta(\theta_{3f})$ are given in solid line in Fig.8, as function of the R . The cyclic motion is stable for $R < 0.8$.

In order to determine if the radius $R = 0.8$ is a limit of stability only for one specific reference path or if this limit is more physical, different kinds of reference motion are considered in the following. Only the interlink angle $h_1(\theta_{3f})$ at the impact is changed. For different values of h_1 and radii R , the coefficient involves in the convergence condition are drawn in Fig.8.

$\delta_{\dot{\theta}_3}$ and δ_I increase when R increases and $h_1(\theta_{3f})$ decreases from Fig.8. δ^2 also increases at the same time. The term δ^2 comes to unit when $R = 0.8 \text{ [m]}$ which means that R has the same values as the length of legs at the impact.

Remark: We confirmed in another simulations that variations of the torso angle had small influences on δ_I and $\delta_{\dot{\theta}_3}$ although it essentially affects ζ . The variables V , δ_{SS} , I_{θ_3} and ζ in the analytic solution for SS phase change for the torso angle. However the variation of δ_{SS} is smaller than the variations of δ_I and $\delta_{\dot{\theta}_3}$ with respect to the foot radii. \triangle

Fig.9 presents the stability property with respect to the foot radii. Two black rigid lines show V and $\delta^2 - 1$. V and $\delta^2 - 1$ have opposite sign thus a cyclic motion may exist such that (45) is satisfied for any value of radii R . For $R < 0.8 \text{ [m]}$, the motion is stable. For $R > 0.8 \text{ [m]}$, the

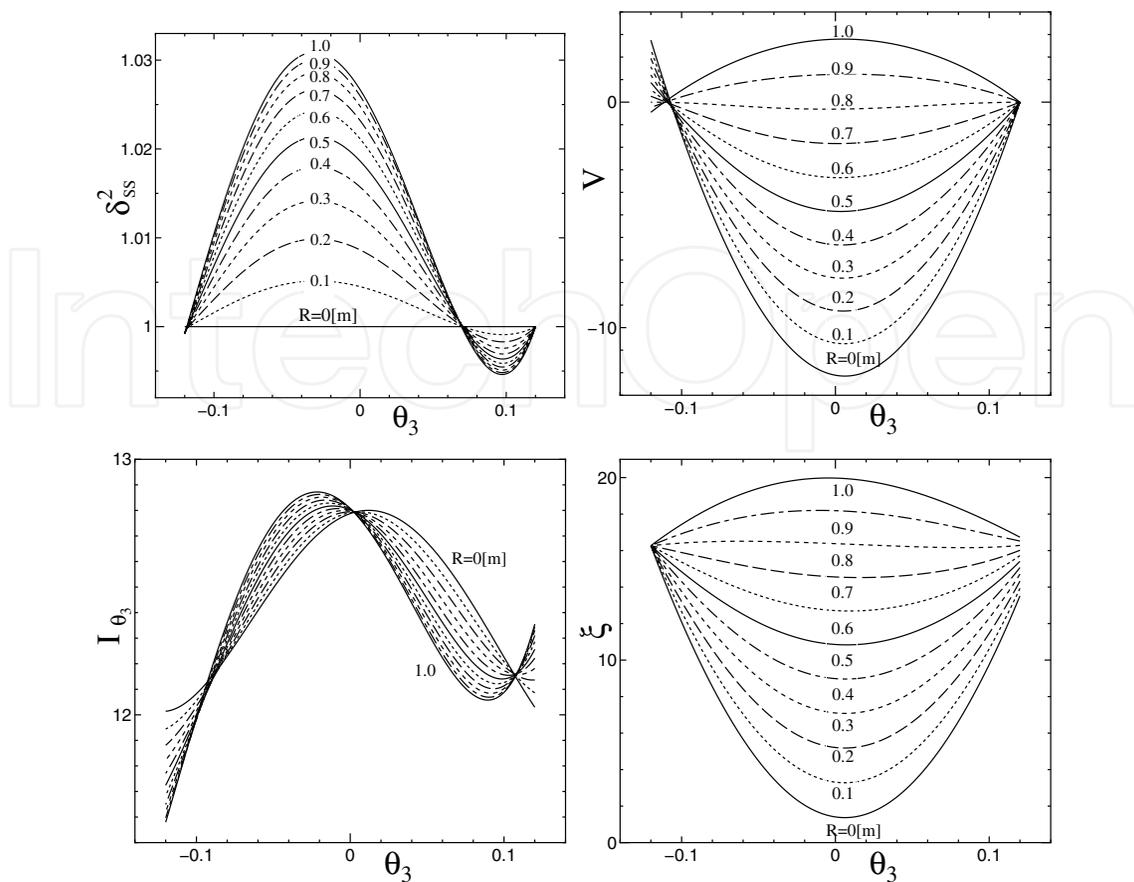


Fig. 7. Analytic solutions for SS phase. The figures are δ_{SS}^2 by Eq. (36) (above left), V by Eq. (37) (above right), the function I_{θ_3} by Eq.(31) (bellow left) and ζ by Eq. (35) (bellow right). θ_3 evolves from positive (0.12) to negative (-0.12).

motion is unstable. For $R = 0.8$ [m], the motion is neutral, in this case any value ζ_c produces cyclic motions.

Case corresponding to a radius superior to the length of each leg, ($R > 0.8$ [m]) can be studied if we consider that the motions of feet are not in the same sagittal plane to avoid collisions. In the leg exchange, at the impact, the contact point moves back but the contact point has a large forward progression during the single support phase, the biped goes forward.

The gradient δ^2 (Eq. (43)) of Poincaré map (Eq. (42)) depends on the SS phase (δ_{SS}) and the impact phase ($\delta_I \cdot \delta_{\dot{\theta}_3}$). δ_{SS} was close to unit at the impact. Since $\dot{y}_G^- < \dot{y}_G^+ < 0$ (see Fig.6), we obtain that the foot radius R and the sign of d defined the position of the ratio $\delta_I \delta_{\dot{\theta}_3}$ with respect to 1 from Eq. (54) to Eq. (56).

- if $R < l$, $d > 0$, and $\delta_I \delta_{\dot{\theta}_3} < 1$
- if $R = l$, $d = 0$, and $\delta_I \delta_{\dot{\theta}_3} = 1$
- if $R > l$, $d < 0$, and $\delta_I \delta_{\dot{\theta}_3} > 1$

The property of the gradient δ^2 agrees with “speed mode” of passive dynamic walking obtained by McGeer McGeer (1990). Wisse Wisse et al. (2006) finds results that are different from our results. For passive walking he finds that for stability point of view the best radius is 14% of leg length, this value corresponds to a case where two monotonic lines of eigenvalues are crossing. The increasing one is represented ‘Speed mode’, and the decreasing one is

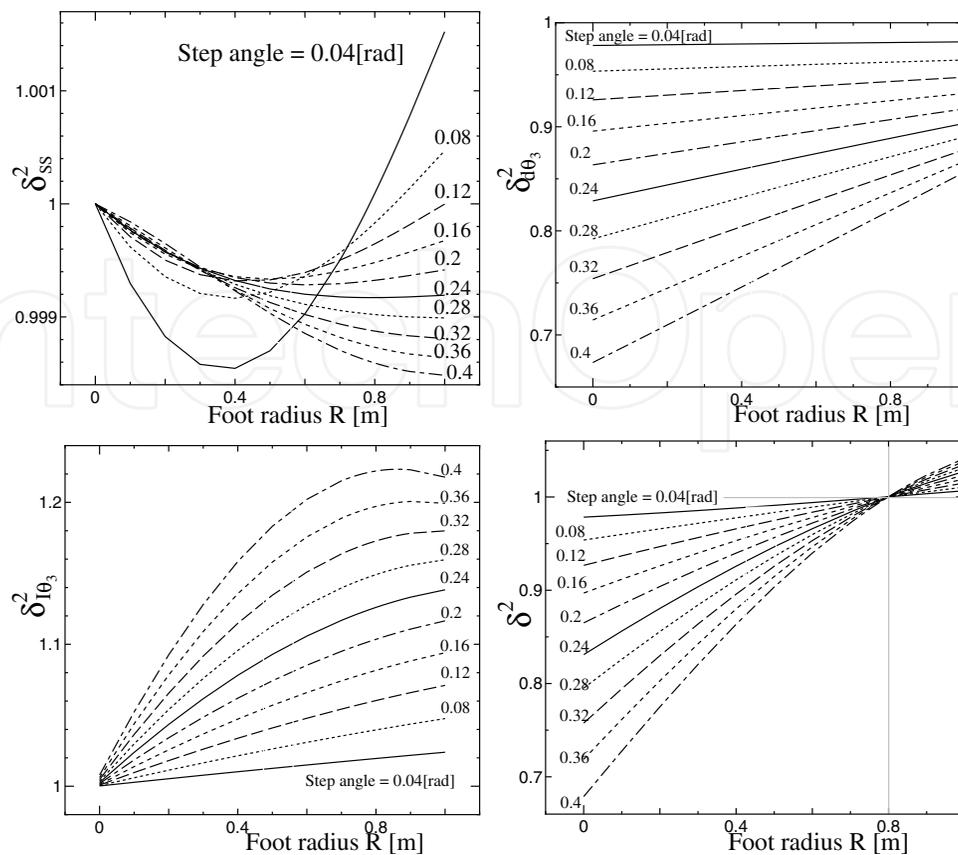


Fig. 8. Slope of the Poincaré return map. Step angle which means angles between two legs at the impact varies from 0.04 [rad] to 0.40 [rad]. The figures show δ_{SS}^2 (above left), $\delta_{\theta_3}^2$ (above right), $\delta_{\theta_3}^2$ (bellow left) and δ^2 with respect to the foot radii $R = 0 \sim 1.0$ [m] (bellow right). $R = 0.8$ [m] means that the radius is the same as the leg length at the impact for the analytic solution. For $R = 0.8$ [m], the cyclic motion is not stable.

'Totter mode'. However the crossing point changes with respect to slope angle and physical parameters of bipeds. The 14% of leg length is not the best radius, generally speaking. In our controlled system, it is predictable that the 'Totter mode' is close to zero or much smaller than the 'Speed mode', since the 'Speed mode' is expressed by the zero dynamics of the controlled system and the 'Totter mode' is depending on the controller gains. Term δ^2 has the same property of the 'Speed mode', and thus is increasing with respect to R . In our case we are not interested in the best solution but in the limit where stability exists, thus there are no contradiction with the results of Wisse Wisse et al. (2006).

6.3 Basin of attraction

Basins of attraction determined by numerical computations are shown in Fig.10. The larger the foot radii are in the stable domain, the wider the basin of attraction is but the slower the speed of convergence is. If the foot radius is the same as the leg length, the motion is neutral, that is, all motions are cyclic.

In Fig.10, the area between the line of ζ_{\min}^- and ζ_{\max}^- is the basin of attraction. The variable ζ just before the impact is used for expressing the basin of attraction. The line ζ_c represents the cyclic motions. Fig.11 presents time evolutions of θ_3 , $\dot{\theta}_3$ for 100 steps. The following foot radii are considered: $R = 0$ [m], 0.5 [m], 0.8 [m] and 1.3 [m]. The first two cases are clearly

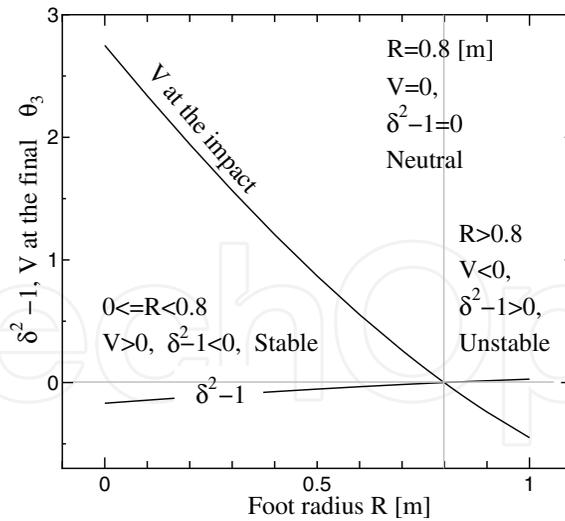


Fig. 9. The property of stability with respect to the foot radii R . Two black rigid lines show V and $\delta^2 - 1$. V and $\delta^2 - 1$ have opposite sign thus a cyclic motion may exist such that (45) is satisfied. For $R < 0.8$ [m] the motion is stable. For $R > 0.8$ [m] the motion is unstable. For $R = 0.8$ [m] the motion is neutral, that is all of ζ_c are cyclic motions.

stable, the case, $R = 0.8$, is neutral, and the case, $R = 1.3$, is unstable. Simulations confirm the existence of the neutral condition.

The property of the basin of attraction with respect to the radius is also analogous to the results of passive dynamic walking by Wisse Wisse & van Frankenhuyzen (2003). As depicted in Fig.10, the bottom line shows minimal ζ corresponding to ζ_{min} . It means a required minimal angular momentum to overcome a gap from a minimum of a vertical position of CoM to a maximum. If the momentum is smaller than the minimum, the complete step is not achieved, the step begins and then the robot goes backward to return to its initial configuration for the step. After that, the robot stops, but it does not fall down contrarily to a passive dynamic walker Wisse et al. (2005) that falls down backward.

From Fig.5, the smaller the radius is, the larger the gaps of the vertical positions of CoM and the minimal ζ_{min} are. Thus the circular arc feet broaden the minimal bounds. The variation of the maximal bounds is caused by limits on the vertical reaction forces to avoid taking-off. The reaction force vector R_1 at the point N_1 is given by the following equation:

$$R_1 = \begin{bmatrix} R_{x_1} \\ R_{y_1} \end{bmatrix} = \begin{bmatrix} M\ddot{x}_G \\ M(\ddot{y}_G + g) \end{bmatrix}. \quad (60)$$

The vertical acceleration \ddot{y}_G is decided by the the centrifugal force caused by the angular velocity of the stance leg $\dot{\theta}_3$ and an acceleration of the leg variation $\ddot{l}_i(t)$. The radius smoothens the variation of CoM, and consequently the centrifugal force is reduced. We observe that the acceleration of the leg is smaller when the radii increase. Thus, the maximal ζ_{max} is extended when the radius increases. Namely, the basin of attraction is broaden by physical properties such as the feet radii. Globally, our controlled system has similar properties for stability and basin of attraction to the passive dynamic walking.

6.4 Consumed energy

Consumed energies and specific resistance for one cyclic step with respect to the foot radii R are described in Fig.12. The following formula is used for computing the consumed energy

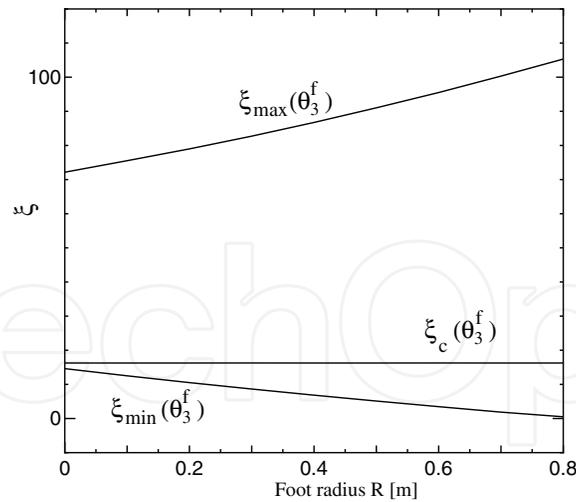


Fig. 10. Basin of attraction of ζ w.r.t. the foot radii R . The area between the line of $\zeta_{\min}(\theta_3^f)$ and $\zeta_{\max}(\theta_3^f)$ is the basin of attraction by the numerical method. The line ζ_c means the cyclic motions. In the upper area of $\zeta_{\max}(\theta_3^f)$, vertical reaction forces are negative. There would be a flight phase. In the lower area of $\zeta_{\min}(\theta_3^f)$, the velocity of the monotonic variable after the impact is not large enough to produce a step, $\zeta_{\min}(\theta_3^f)$ is given by (45). After the beginning of the step, the biped goes backward or stands still eventually.

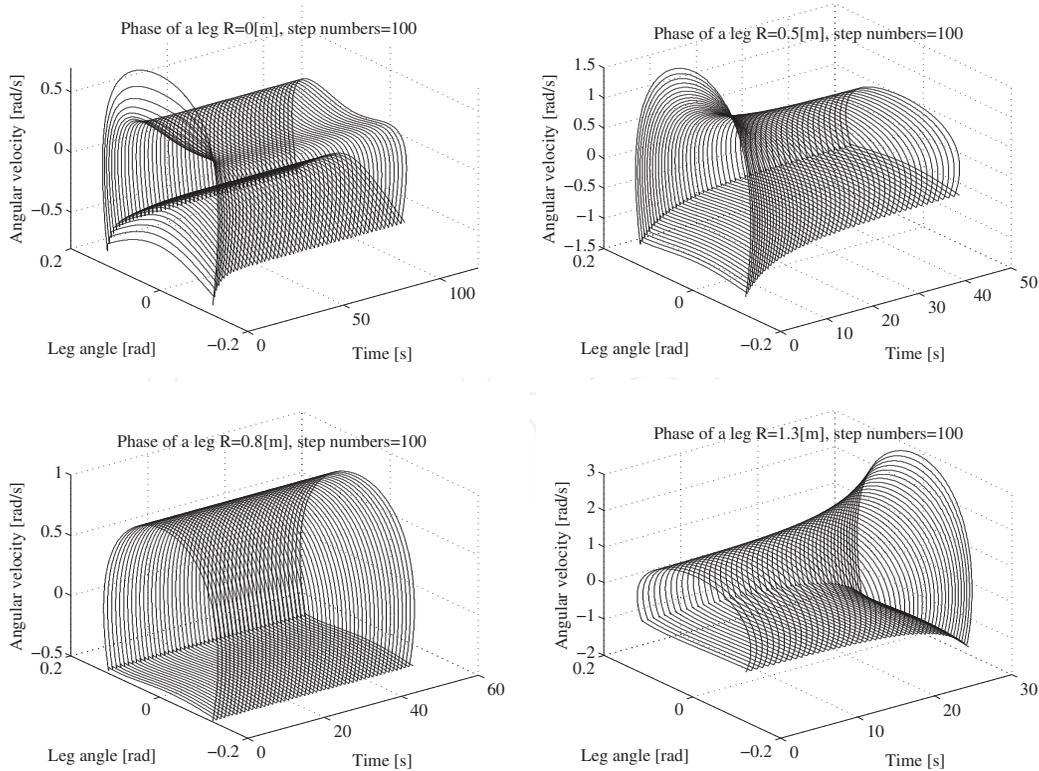


Fig. 11. Time evolutions of phases for the first leg at the foot radii $R = 0$ [m] (stable, above left), 0.5 [m] (stable, above right), 0.8 [m] (neutral, below left) and 1.3 [m] (unstable, below right).

E_c :

$$E_c = \int_0^T |\dot{\theta}' \cdot B \cdot \Gamma| dt. \quad (61)$$

The specific resistance SR is computed by the following fomula:

$$SR = \frac{E_c}{Mgd_{x_G}} \quad (62)$$

d_{x_G} indicates distance of total CoG for one step in horizontal direction. The larger the foot radius is, the smaller the consumed energy as well as the specific resistance is for the cyclic motion, even if the motion becomes unstable. Thus, the circular arc feet are effective in reducing the consumed energy.

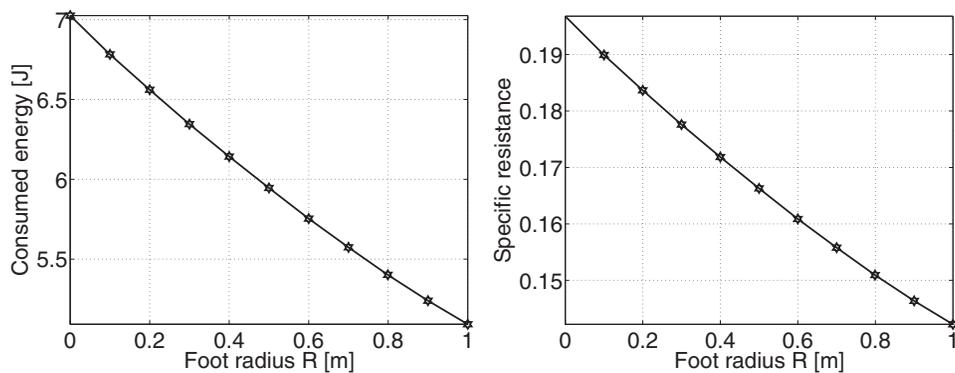


Fig. 12. Consumed energy (left) and specific resistance (right) for one cyclic step w.r.t. the foot radii R by the numerical simulation. The torso angle is chosen so that $\zeta = 16.27$ by the analytic solution for all R .

6.5 Optimal radius

There is a trade-off property between the convergence speed, the basin of attraction and the energy consumption. What we can say is that the nearer the radius is to the leg length, the slower the speed of convergence is and the larger the basin is. 'Foot clearance problem' does not appear because of the variable length legs in our case. In the cases of 'Anthropomorphic Model' and 'Simplest Model' of Adamczyk's result Adamczyk et al. (2006), the CoM mechanical work property with respect to feet radii is similar to our result of consumed energy. However, in their cases of 'Forward-foot Model' and 'Kneel Model', the work had a minimum.

The suggestion of McGeer's to choose a foot radius of 1/3 of leg lengths can also be considered in our discussion. It might be better to choose a larger radius (e.g. between a half and three quarters) to have a large basin of attraction even if the speed of convergence is worth.

6.6 Unstable walking with radii greater than the leg length

Kuo's analysis Kuo (2001) of the CoM velocity contradicts our study because he considers a simple model with rigid legs and circular arc feet and the CoM is located at hip position, and we consider prismatic knees. The right of Fig.5 presents the evolution of the CoM relative to the simple model Kuo (2001). Tangent vectors of right ends of lines are expressing the pre-impact velocity of CoM, and tangent vectors of left ends of lines are expressing the

post-impact velocity of CoM. When $R > 0.8$ [m], the change of CoM velocities are upward, which means the impulsive force at the impact is negative. It actually would be a flight phase. Left part of Fig.5 gives the CoM evolution in the case of our biped shown in Fig.2. Since all of the ranges of velocities of CoM at the impact are downward, it never fails to flight phase for any radius. In fact, our biped has prismatic knees and CoM is mainly distributed on the torso which is swinging a little. A lot of paths can be chosen for the CoM position differently from the simple model.

7. Conclusion

In the paper, some effects of circular arc feet for a planar biped via a geometric tracking were taken into account. An analytic solution of Poincaré map was given for the controlled system. Stability of walking was analyzed by the Poincaré map and the following results are obtained:

- Radii of the circular arc feet affect the stability of walking, and the speed of convergence decreases when the radii approaches to a leg length.
- A basin of attraction is broadened by choosing larger radii and the controller can stabilize the biped walking in the largest basin of attraction for the radii less than the leg length.

The leg length and the radius smoothen the variation and reduce the impact velocity. From the properties of the reference paths, The radius of the foot has a significant effect for the stability and the basin of attraction. The results are analogous to those McGeer (1990); Wisse & van Frankenhuyzen (2003) and the prospect Wisse et al. (2005) on passive dynamic walking. The geometric tracking method does not change the general effect of the circular arc feet. A reduction of the vertical CoM variation by the foot radius is functional not only for the geometric tracking method but for general biped walking. However the motion of CoM and the consumed energy are different from some very simple models because our model has variable length of legs and a torso.

8. References

- Aoustin, Y. & Formalsky, A. M. (1999). Design of Reference Trajectory to Stabilize Desired Nominal Cyclic Gait of a Biped, *Proceedings of the International Workshop on Robot Motion and Control, ROMOCO'99*, IEEE, Kiekrz, Poland, 1999, pp.159-165.
- Grizzle, J.W.; Abba, G. & Plestan, F. (2001). Asymptotically stable walking for biped robots: Analysis via systems with impulse effects, *IEEE transaction on Automatic Control*, Vol. 46, No, 1, (Jan. 2001), 51-64, ISSN 0018-9286.
- Aoustin, Y. & Formalsky, A. M. (2003). Control Design for a Biped Reference Trajectory Based on Driven Angles as Functions of the Undriven Angle, *Journal of Computer and System Sciences International*, Vol.42, No.4, (Apr. 2003), pp.159-176, ISSN 1064-2307.
- Chevallereau, C.; Abba, G.; Aoustin, Y.; Plestan, Y.; Westervelt, E.R.; Canudas-De-Wit, C. & Grizzle, J.W. (2003). Rabbit: A test bed for advanced control theory, *IEEE Control System Magazine*, Vol.23, No.5, (Oct. 2003), pp.57-79, ISSN 0272-1708.
- Furusho, J.; Moritsuka, H. & Masubuchi, M. (1981) Low Order Modeling of Biped Locomotion System Using Local Feedback, *Transactions of the Society of Instrument and Control Engineers*, Vol.17, No.5, (May 1981), pp.596-601, ISSN 0453-4654
- Kajita, S. & Tani, K. (1991) Study of Dynamic Biped Locomotion on Rugged Terrain – Derivation and Application of the Linear Inverted Pendulum Mode –, *Proceedings of IEEE International Conference on Robotics and Automation, ICRA1991*, IEEE, California, USA, Apr-1999, pp.1405-1411.

- Westervelt, E.R.; Buche, G. & Grizzle, J.W. (2004) Experimental Validation of a Framework for the Design of Controllers that Induce Stable Walking in Planar Biped, *International Journal of Robotics Research*, Vol.23, No.6, (Jun. 2004), pp.559-582, ISSN 0278-3649.
- Kinugasa, T; Hashimoto, Y & Fushimi, H. (2003) Passive Walking of Biped Emu with Attitude Control of Body, *Proceedings of IEEE/RSJ International Conference on Intelligent Robots and Systems, IROS 2003*, IEEE, Las Vegas, USA, Oct-2003, pp.346-359.
- Kinugasa, T.; Osuka, K. & Miwa, S. (2007) Biped Walking by Variations of Knee Lengths and Attitude Control of a Body and its Frequency Analysis, *Journal of the Robotics Society of Japan*, Vol. 25, No. 3, (Apr. 2007), pp.116-123, ISSN 0289-1824
- Chevallereau, C. & Djoudi, D. (2006) Feet can improve the stability property of a control law for a walking robot, *Proceedings of IEEE International Conference on Robotics and Automation, ICRA2006*, IEEE, Florida, USA, May-2006, pp.1206-1212.
- McGeer, T. (1990) Passive Dynamic Walking, *International Journal of Robotics Research*, Vol.9, No.2, (Feb. 1990), pp.62-82, ISSN 0278-3649.
- Wisse, M. & van Frankenhuyzen, J., (2003) Design and construction of mike; a 2d autonomous biped based on passive dynamic walking, *Proceedings of Conference on Adaptive Motion of Animals and Machines*, Kyoto, Japan, Mar-2003, WeP-I-1.
- Wisse, M.; Schwab, A.L.; van der Linde, R.Q. & van der Helm, F.C.T., (2005) How to Keep From Falling Forward: Elementary Swing Leg Action for Passive Dynamic Walkers, *IEEE Transactions on robotics*, Vol.21, No.3, (May 2005), pp.394-401, ISSN 1552-3098.
- Wisse, M.; obbelen, D.G.E.; Rotteveel, R.J.J.; Anderson, S.O. & Zeglin, G.J.,(2006) Ankle Springs Instead of Arc-shaped Feet for Passive Dynamic Walkers, *Proceedings of IEEE-RAS International Conference on Humanoid Robots 2006*, Genova, Italy, Dec-2006, pp.110-116.
- Asano, F. & Zhi-Wei Luo, (2007) The Effect of Semicircular Feet on Energy Dissipation by Heel-strike in Dynamic Biped Locomotion, *Proceedings of IEEE International Conference on Robotics and Automation 2007*, Rome, Italy, Apr-2007, pp.3976-3981.
- Adamczyk, P. G.; Collins, S. H. & Kuo, A. D., (2006) The advantages of a rolling foot in human walking, *The Journal of Experimental Biology* 209, (Oct. 2006), pp.3953-3963, ISSN 0022-0949.
- Morris, B. & Grizzle, J. W., (2005) A Restricted Poincare Map for Determining Exponentially Stable Periodic Orbits in Systems with Impulse Effects: Application to Bipedal Robots, *Proceedings of IEEE Conference on Decision and Control 2005*, Seville, Spain, Dec-2005, pp.4199-4206.
- Kuo, A. D., (2001) Energetics of Actively Powered Locomotion Using the Simplest Walking Model, *Journal of Biomechanical Engineering*, 124, (2001), pp.113-120, ISSN 0148-0731.
- Chevallereau, C.; Formal'sky, A. & Djoudi, D., (2004) Tracking of a Joint Path for the Walking of an Underactuated Biped, *Robotica*, vol.22, (2004), pp.15-28, ISSN 0263-5747.
- Farrell, K. D.; Chevallereau, C. & Westervelt, E. R. (2007) Energetic Effects of Adding Springs at the Passive Ankles of a Walking Biped Robot, *Proceedings of IEEE International Conference on Robotics and Automation, Roma, Italy, Apr-2007*, pp.3591-3596, ISSN 1050-4729.



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Biped robots represent a very interesting research subject, with several particularities and scope topics, such as: mechanical design, gait simulation, patterns generation, kinematics, dynamics, equilibrium, stability, kinds of control, adaptability, biomechanics, cybernetics, and rehabilitation technologies. We have diverse problems related to these topics, making the study of biped robots a very complex subject, and many times the results of researches are not totally satisfactory. However, with scientific and technological advances, based on theoretical and experimental works, many researchers have collaborated in the evolution of the biped robots design, looking for to develop autonomous systems, as well as to help in rehabilitation technologies of human beings. Thus, this book intends to present some works related to the study of biped robots, developed by researchers worldwide.

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