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Calculation Methods for Thermoelectric Generator Performance

Fuqiang Cheng

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Abstract

This chapter aims to build one-dimensional thermoelectric model for device-level thermoelectric generator (TEG) performance calculation and prediction under steady heat transfer. Model concept takes into account Seebeck, Peltier, Thomson effects, and Joule conduction heat. Thermal resistances between heat source, heat sink, and thermocouple are also considered. Then, model is simplified to analyze influences of basic thermal and electrical parameters on TEG performance, when Thomson effect is neglected. At last, an experimental setup is introduced to gauge the output power and validate the model. Meantime, TEG simulation by software ANSYS is introduced briefly.

Keywords: thermoelectric generator, thermoelectric model, output power, thermoelement

1. Introduction

Output power P_{out} and energy conversion efficiency η are the primary parameters to characterize TEG performance. They are intensively influenced by such factors as temperature of heat source and sink, thermoelectric materials physical properties, thermocouple geometries, thermal and electrical contact properties, and load factor. Therefore, it is necessary to build physical model formulating these factors concisely, to conduct realistic TEG design. At present, many significant works have been undertaken for modeling device-level TEG precisely [1–3]. In addition, comprehensive three-dimensional (3D) thermoelectric model has been successfully developed in software ANSYS [4]. In Refs. [5–7], quasi-one-dimensional thermoelectric model is established, where Thomson effect and thermal resistances between thermocouple and heat source, heat sink are neglected. In Ref. [8], improved one-dimensional model

including Thomson coefficient and thermal resistances is used to analyze the matched load, the limit of energy conversion efficiency, and the influence of Peltier effect. It shows, that expression of matched load contains not only the inner electrical resistance of TEG, but also the terms resulting from Peltier and Joule effects. In Ref. [9], one-dimensional model to analyze the influence of Thomson heat is built and experimentally validated.

In this chapter, Seebeck, Peltier, Thomson effect, and Joule conduction heat are formulated in thermoelectric generation module model. By model simplification, analytical expressions of output power and energy efficiency are introduced. Essential factors for enhancing the output power are extracted. Then, an experimental setup is built to measure the output power and validate the model. And TEG simulation by software ANSYS is presented.

2. Thermoelectric model for device-level TEG

2.1. TEG cell structure

TEG cell consisting of thermocouple is shown in **Figure 1**, where basic thermoelectric effects including Peltier and Joule heat and a circuit with load R_L are included. The p and n thermoelements are cuboids of the same thickness and bridged by an electrode in series. Practical devices usually make use of thermoelectric modules containing a number of TEG cells connected electrically in series and thermally in parallel. Cross-sectional area and thickness of thermocouple are marked as A and l . Subscripts 'n' and 'p' are used to discriminate conductivity type of thermoelements. Temperature of heat source and heat sink is T_1 and T_0 , and that of hot and cold side of thermocouple is T_h and T_c . $\Delta T_g = T_h - T_c$ is temperature difference on thermoelements, and $\Delta T = T_1 - T_0$ is the one of heat source and heat sink.

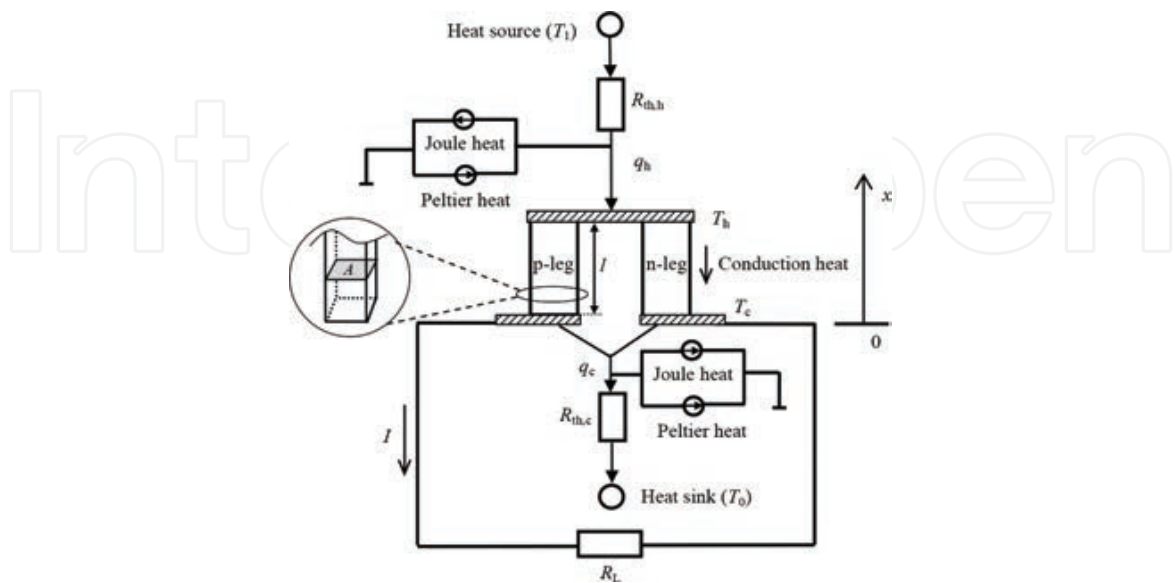


Figure 1. Structure and circuit sketch of TEG cell.

There are Joule heat flowing out and Peltier heat flowing in at hot end of thermoelements, and at cold end, Peltier heat flows out and Joule heat flows out. In addition, there is thermal resistance $R_{th,h}$ and $R_{th,c}$ between thermoelements, heat source and heat sink. Heat flow q_h passes from heat source to hot side of thermocouple and the counterpart q_c outflows from cold side of thermocouple to heat sink.

2.2. Basic model

It is assumed, that thermoelements are physically homogeneous and insulated from the surroundings both electrically and thermally, except at junction-reservoir contacts [8–9]. Variable x is defined as location in the thickness direction of thermoelements. According to nonequilibrium thermodynamics under steady heat transfer, energy conservative equations of temperature distributions $T_n(x)$ and $T_p(x)$ are:

$$\begin{cases} K_n l_n \frac{d^2 T_n(x)}{dx^2} - \tau_n I \frac{dT_n(x)}{dx} + \frac{R_n I^2}{l_n} = 0 \\ K_p l_p \frac{d^2 T_p(x)}{dx^2} + \tau_p I \frac{dT_p(x)}{dx} + \frac{R_p I^2}{l_p} = 0 \end{cases} \quad (1)$$

Three terms in the above equations represent thermal conduction, Thomson and Joule heat. K , R , and τ are thermal conductance, electrical resistance and Thomson coefficient ($V \cdot K^{-1}$), respectively. Relationship of K , R and A , l is $K = \frac{\lambda A}{l}$ and $R = \frac{\rho l}{A}$, where λ and ρ are thermal conductivity and electrical resistivity of thermoelectric materials. To solve Eq. (1) analytically, material parameters K , R , and τ are considered to be constant. The boundary conditions of Eq. (1) are:

$$T_n(0) = T_p(0) = T_c, \quad (2)$$

$$T_n(l_n) = T_p(l_p) = T_h. \quad (3)$$

Electrical current I is determined by formula:

$$I = \frac{U_0}{R_L + R_g}, \quad (4)$$

$$U_0 = \int_{T_c}^{T_h} \alpha(T) dT, \quad (5)$$

where U_0 is the voltage of thermocouple, R_g is the electrical resistance of TEG cell, which contains resistance of thermocouple and contact resistance, and $\alpha(T) = \alpha_p(T) - \alpha_n(T)$ is Seebeck coefficient ($V \cdot K^{-1}$) of thermocouple.

In practice, temperature of heat source T_1 and heat sink T_0 can be measured and determined. To acquire T_h and T_c , relationship of T_1 , T_0 and T_h , T_c is necessary. That is:

$$\begin{cases} T_1 - T_h = R_{th,h} q_h \\ T_c - T_0 = R_{th,c} q_c \end{cases} \quad (6)$$

In Eq. (6), heat flows q_h and q_c are:

$$q_h = K_n l_n \frac{dT_n(x)}{dx} \Big|_{x=l_n} + K_p l_p \frac{dT_p(x)}{dx} \Big|_{x=l_p} + \alpha(T_h) T_h I - I^2 R_{ch}, \quad (7)$$

$$q_c = K_n l_n \frac{dT_n(x)}{dx} \Big|_{x=0} + K_p l_p \frac{dT_p(x)}{dx} \Big|_{x=0} + \alpha(T_c) T_c I + I^2 R_{cc}, \quad (8)$$

wherein R_{ch} and R_{cc} are contact electrical resistances at hot and cold side of the thermocouple. Thermal conduction heat, Peltier heat (the third term), and contact Joule heat are within Eqs. (7) and (8). By solving Eq. (1) with Eqs. (2)–(5), $T_n(x)$ and $T_p(x)$ only relating to T_h , T_c and R_L can be obtained. And flows q_h and q_c can be formulated with T_h , T_c and R_L in Eqs. (7) and (8). Then, T_h and T_c can be determined for a given R_L by solving Eq. (6) numerically, which is presented in detail [9].

Finally, the output power P_{out} and energy conversion efficiency η are calculated by the basic equations of thermoelectricity:

$$P_{out} = q_h - q_c = I^2 R_L, \quad (9)$$
$$\eta = \frac{P}{q_h}. \quad (10)$$

When neglecting Thomson heat, the problem will be much simplified. By solving Eqs. (1)–(8) with $\tau = 0$, an cubic equation about ΔT_g can be yielded as:

$$a_1 \Delta T_g^3 + b_1 \Delta T_g^2 + c_1 \Delta T_g + 1 = 0, \quad (11)$$

where:

$$a_1 = \frac{\bar{\alpha}^4 R_L R_{th,c} R_{th,h}}{(R_g + R_L)^3 \Delta T},$$

$$b_1 = -\frac{\bar{\alpha}^2 R_{th,c} R_{th,h}}{(R_g + R_L)^2 \Delta T} \left[R_g \left(\frac{\varepsilon}{R_{th,c}} + \frac{\varepsilon - 1}{R_{th,h}} \right) + R_L \left(\frac{1}{R_{th,c}} - \frac{1}{R_{th,h}} \right) \right],$$

$$c_1 = -\frac{R_{th,c} R_{th,h}}{\Delta T} \left[\left(\frac{T_1}{R_{th,h}} + \frac{T_0}{R_{th,c}} \right) \frac{\alpha^2}{R_g + R_L} + \frac{K_g}{R_{th,h}} + \frac{K_g}{R_{th,c}} + \frac{1}{R_{th,c} R_{th,h}} \right],$$

$$\varepsilon = \frac{0.5R_0 + R_{cc}}{R_g},$$

and Seebeck coefficient α becomes a constant. Equation (11) suits thermoelectric module consisting of m thermocouples, as well, where α and K_g are m times of those of a single thermocouple, but $R_{th,h}$ and $R_{th,c}$ are exactly on the contrary.

Generally, c_1 is far larger than a_1 and b_1 in absolute value. Mainly, because of practical module, Seebeck coefficient α has a very small value of about $10^{-2} \text{ V} \cdot \text{K}^{-1}$, which is much less than unity. For example, taking module TEG-127-150-9 in Ref. [8], $\alpha = 0.05 \text{ V} \cdot \text{K}^{-1}$, $R_g = 3.4 \text{ Ohm}$, $R_L = 4 \text{ Ohm}$, $R_{th,c} = 6 \text{ K} \cdot \text{W}^{-1}$, $R_{th,h} = 0.1 \text{ K} \cdot \text{W}^{-1}$, $K_g = 2.907 \text{ W} \cdot \text{K}^{-1}$, $\varepsilon \approx 0.5$, $T_0 = 297 \text{ K}$, and $T_1 = 323 \text{ K}$, calculation result is $a_1 \approx 1.423 \times 10^{-9} \text{ K}^{-3}$, $b_1 \approx 5.905 \times 10^{-5} \text{ K}^{-2}$, and $c_1 \approx -0.7461 \text{ K}^{-1}$. So, the terms with ΔT_g order higher than unity can be neglected. At last, here is:

$$\Delta T_g = \frac{\Delta T}{1 + R_{th,c} K_g + R_{th,h} K_g + \alpha^2 \frac{(R_{th,c} T_1 + R_{th,h} T_0)}{R_g + R_L}}. \quad (12)$$

It can be seen, that ΔT_g is influenced not only by thermal resistances $R_{th,c}$ and $R_{th,h}$ but also by Peltier effect, which is presented in the last term of the denominator and functions to decrease ΔT_g . Because it is tantamount to accelerate heat conduction in thermocouple, Peltier heat flows in and out on two sides of thermocouple. By combining Eqs. (12) and (4), (5), (9), the output power P_{out} is:

$$P_{out} = \frac{\alpha^2 \Delta T_g^2 R_L}{(R_g + R_L)^2} = \frac{\alpha^2 \Delta T^2 R_L}{(1 + R_{th,c} K_g + R_{th,h} K_g)^2 \left[R_g + R_L + \frac{\alpha^2 (R_{th,c} T_1 + R_{th,h} T_0)}{1 + R_{th,c} K_g + R_{th,h} K_g} \right]^2}. \quad (13)$$

$R_L, T_1, T_0, R_{th,c}, R_{th,h}, \alpha, R_g$, and K_g directly affect P_{out} . In those parameters, α, R_g and K_g are TEG internal factors, and R_L, T_1 , and T_0 are the external ones, and $R_{th,c}$ and $R_{th,h}$ originate from both the internal and external. From the form of Eq. (13), it is obvious, that reducing $T_1, T_0, R_{th,c}$ and $R_{th,h}$ can increase P_{out} if ΔT is constant, owing to influence of Peltier effect on ΔT_g . On the other hand, P_{out} has a maximum along with R_g and K_g .

2.3. Matched load, output power and energy efficiency

First of all, influence of R_L on P_{out} is analyzed. In Eq. (13), P_{out} reaches maximum, when R_L is:

$$R_L = R_g + \frac{\alpha^2 (R_{th,c} T_1 + R_{th,h} T_0)}{1 + R_{th,c} K_g + R_{th,h} K_g}, \tag{14}$$

which is the matched load and marked as $R_{L,m}$. Indeed, $R_{L,m}$ is slightly larger than R_g due to the very small value of α^2 . It means, that existence of Peltier effect increases irreversible heat in thermoelectric module. And reducing T_1 and T_0 helps to cut down this irreversible heat. When $K_g \rightarrow +\infty$, $R_{L,m}$ is equal to R_g , since at this moment heat conduction in thermocouple runs under infinitesimal temperature difference and the irreversibility of heat transfer disappears. However, this irreversibility exists with finite K_g , leading to heat loss in thermocouple, that is equivalent to increase in internal resistance. Define R_L/R_g as the load factor s_L . So, s_L is:

$$s_L = 1 + \frac{Z K_g (R_{th,c} T_1 + R_{th,h} T_0)}{1 + R_{th,c} K_g + R_{th,h} K_g}, \tag{15}$$

when R_L is equal to matched load and $Z = \frac{\alpha^2}{K_g R_g}$ is the figure of merit. For thermoelectric module, the output power is:

$$P_{out} = \frac{m^2 \alpha^2 \Delta T^2 R_L}{(1 + R_{th,c} K_g + R_{th,h} K_g)^2 \left\{ R_L + m \left[R_g + \frac{\alpha^2 (R_{th,c} T_1 + R_{th,h} T_0)}{1 + R_{th,c} K_g + R_{th,h} K_g} \right] \right\}^2}, \tag{16}$$

where m is the number of thermocouples. And the corresponding matched load is

$$R_{L,m} = m \left[R_g + \frac{\alpha^2 (R_{th,c} T_1 + R_{th,h} T_0)}{1 + R_{th,c} K_g + R_{th,h} K_g} \right].$$

As for energy efficiency η , by Eq. (7), which can be expressed as function of ΔT_g and Eqs. (12) and (13), it is:

$$\eta = \frac{P}{q_h} = \frac{\alpha^2 \Delta T^2 R_L}{c_2^2 (R_L + R_g)^2 \left\{ \frac{\alpha^2 \Delta T \left[T_h (R_L + R_g) - \frac{\varepsilon \Delta T R_g}{c_2} \right]}{c_2 (R_L + R_g)^2} + \frac{\Delta T K_g}{c_2} \right\}}, \quad (17)$$

where

$$c_2 = 1 + R_{th,c} K_g + R_{th,h} K_g + \frac{\alpha^2 (R_{th,c} T_1 + R_{th,h} T_0)}{R_L + R_g}.$$

By solving Eq. (17) about the partial derivative of R_L , it can be obtained, that when load factor s_L is:

$$s_L = \sqrt{1 + ZT_h + \frac{Z\varepsilon\Delta T + ZK_g(1 + ZT_h)(R_{th,c}T_1 + R_{th,h}T_0)}{1 + R_{th,c}K_g + R_{th,h}K_g}}, \quad (18)$$

then η reaches maximum. Equation (18) is downright different from Eq. (15) in the expressions, so achieving maximum of output power and energy efficiency simultaneously is impossible. Actually, the corresponding load factor of the former is smaller than that of the latter. When the ideal state is considered ($R_{th,c} = R_{th,h} = 0$), $s_L = 1$ is for the former and $s_L = \sqrt{1 + ZT_h + Z\varepsilon\Delta T}$, which is larger than 1, is for the latter.

2.4. Influence of K_g on TEG performance

K_g is important internal factor that influences the output performance in TEG. When matched load is reached, the corresponding output power $P_{out,m}$ is:

$$P_{out,m} = \frac{ZK_g\Delta T^2}{4(1 + R_{th,c}K_g + R_{th,h}K_g)^2 \left[1 + \frac{ZK_g(R_{th,c}T_1 + R_{th,h}T_0)}{1 + R_{th,c}K_g + R_{th,h}K_g} \right]}. \quad (19)$$

For a common thermoelectric module, thermoelements have the same size, $l_n = l_p$ and $A_n = A_p$, so the figure of merit Z is not related to their size, but material physical parameters. From Eq. (19), we can see, that increase in Z will enhance the output power. By solving Eq. (19) regarding the partial derivative of K_g , when:

$$K_g = \frac{1}{\sqrt{(R_{th,c} + R_{th,h})^2 + Z(R_{th,c}T_l + R_{th,h}T_0)(R_{th,c} + R_{th,h})}} = (\lambda_p + \lambda_n) \frac{A_e}{l_e}, \tag{20}$$

then $P_{out,m}$ reaches maximum, where l_e and A_e are thickness and cross-sectional area of thermoelements. Since $R_{th,c}$ and $R_{th,h}$ are related to A_e , but not to l_e , there is an optimal l_e to maximize $P_{out,m}$:

$$P_{out,m} = \frac{Z\Delta T^2}{4c_3 \left(1 + \frac{R_{th,c}}{c_3} + \frac{R_{th,h}}{c_3}\right)^2 \left[1 + \frac{Z(R_{th,c}T_l + R_{th,h}T_0)}{c_3 \left(1 + \frac{R_{th,c}}{c_3} + \frac{R_{th,h}}{c_3}\right)}\right]}, \tag{21}$$

and

$$c_3 = \sqrt{(R_{th,c} + R_{th,h})^2 + Z(R_{th,c}^2T_l + R_{th,h}^2T_0 + R_{th,c}R_{th,h}T_l + R_{th,c}R_{th,h}T_0)}.$$

2.5. Influence of Peltier effect on TEG performance

When Peltier effect is neglected, the relation of ΔT_g and ΔT is:

$$\Delta T_g = \frac{R_{th,g}}{R_{th,g} + R_{th,h} + R_{th,c}} \Delta T, \tag{22}$$

and the corresponding output power with matched load $R_L = R_g$ and constant Seebeck coefficient is:

$$P_{out,m} = \frac{(\alpha\Delta T)^2}{4R_g(1 + R_{th,h}K_g + R_{th,c}K_g)}. \tag{23}$$

Meantime, the output power Eq. (13) is for the condition without Peltier effect, and the ratio of Eq. (23) and Eq. (19), η^{Pelt} , reflects influence degree of Peltier effect on the output power:

$$\eta_{Pelt} = \left(1 + \frac{ZT_0}{2} \frac{R_{th,c} + R_{th,h}}{R_{th,g} + R_{th,h} + R_{th,c}} + \frac{Z}{2} \frac{R_{th,c}\Delta T}{R_{th,g} + R_{th,h} + R_{th,c}}\right)^2. \tag{24}$$

It is known, that when $R_{th,g} \ll R_{th,c} + R_{th,h}$, η_{Pelt} is approximately $(1 + 0.5ZT_0 + 0.25Z\Delta T)^2$ with $R_{th,c} \approx R_{th,h}$ and even with $\Delta T \rightarrow 0$, the output power calculated without Peltier effect is more than the output power considering Peltier effect, by over 120% for a common Bi_2Te_3 -based module with $ZT \approx 1$. That means, the influence of Peltier effect must be considered. Similar status is obtained, where the difference is more than 50%, when $R_{th,g} \approx R_{th,c} + R_{th,h}$. On the contrary, when $R_{th,g} \gg R_{th,c} + R_{th,h}$, η_{Pelt} is approximately equal to 1, which means the influence of Peltier effect is negligible. Hence, the smaller the thermal resistance of thermocouple $R_{th,g}$, the stronger is the influence of Peltier effect.

Eventually, basic factors for enhancing TEG output power are summarized as:

1. Enhancing ZT of thermoelectric materials and ΔT , decreasing $R_{th,c}$ and $R_{th,h}$.
2. When ΔT is fixed, lower T_0 and T_1 can reduce irreversible heat to elevate output power.
3. Matched load is a little larger than the inner electrical resistance of TEG.
4. There exists an optimal thermocouple thickness to maximize output power.

3. Test validation

3.1. Materials property

P-type and n-type Bi_2Te_3 -based materials are, respectively, $Bi_{0.5}Sb_{1.5}Te_3$ and $Bi_2Te_{2.85}Se_{0.15}$, which are prepared by mechanical alloy + spark plasma sintering method. Seebeck coefficient and resistivity of the materials are tested by HGTE-II thermoelectric material performance test system (Chinese patent no. ZL200510018806.4) with test temperature up to 1073 K, relative error of not more than 6%. Thermal conductivity of the materials is measured by laser perturbation method (Type TC-7000 of ULVAC RIKO®). As shown in **Table 1**, parameters are obtained by polynomial fitting of the experimental data. In temperature range $273\text{ K} < T < 493\text{ K}$, Seebeck coefficient value α is between $170 \times 10^{-6}\text{ V}\cdot\text{K}^{-1}$ and $220 \times 10^{-6}\text{ V}\cdot\text{K}^{-1}$, decreasing with rising temperature. Electrical resistivity ρ is $(8.3\text{--}20.0) \times 10^{-6}\text{ Ohm}\cdot\text{m}$ and thermal conductivity λ is $1.4\text{--}2.1\text{ W}\cdot\text{m}^{-1}\cdot\text{K}^{-1}$, which both show obvious increase with temperature rise.

Material	Parameters	Values
p- $Bi_{0.5}Sb_{1.5}Te_3$	$\alpha/\text{V}\cdot\text{K}^{-1}$	$-1.791 \times 10^{-11}T^3 + 1.763 \times 10^{-8}T^2 - 5.714 \times 10^{-6}T + 8.304 \times 10^{-4}$
	$\rho/\text{Ohm}\cdot\text{m}$	$-7.929 \times 10^{-13}T^3 + 7.992 \times 10^{-10}T^2 - 1.947 \times 10^{-7}T + 1.728 \times 10^{-5}$
	$\lambda/\text{W}\cdot\text{m}^{-1}\cdot\text{K}^{-1}$	$3.342 \times 10^{-5}T^2 - 2.24 \times 10^{-2}T + 5.118$
n- $Bi_2Te_{2.85}Se_{0.15}$	$\alpha/\text{V}\cdot\text{K}^{-1}$	$1.321 \times 10^{-11}T^3 - 1.383 \times 10^{-8}T^2 + 4.81 \times 10^{-6}T - 7.774 \times 10^{-4}$
	$\rho/\text{Ohm}\cdot\text{m}$	$-7.618 \times 10^{-13}T^3 + 8.098 \times 10^{-10}T^2 - 2.537 \times 10^{-7}T + 3.207 \times 10^{-5}$
	$\lambda/\text{W}\cdot\text{m}^{-1}\cdot\text{K}^{-1}$	$3.264 \times 10^{-5}T^2 - 2.228 \times 10^{-2}T + 5.302$

Table 1. Physical parameters of Bi_2Te_3 -based materials ($273\text{ K} < T < 493\text{ K}$).

In practice, it is difficult to measure thermal resistances $R_{th,c}$ and $R_{th,h}$ and contact electrical resistances r_{cc} and r_{ch} . Their values are determined according to empirical formulas. Contact electrical resistivity ρ_c (Ohm·m²) at leg-strap junctions and thermal conductivity λ_c (W m⁻¹ K⁻¹) of thermal conductive layer (≈ 1.2 mm thick) are according to the empirical formulas given by Rowe et al. [5]:

$$\frac{2\rho_c}{\rho} = 0.1 \text{ mm}, \frac{\lambda}{\lambda_c} = 0.2. \tag{25}$$

Here ρ and λ are electrical resistivity and thermal conductivity of thermoelements, respectively. In our calculation, the mean values of ρ and λ over the temperature range are taken as references. As ρ and λ vary with temperature, values of ρ_c and λ_c are also different as the temperature varies. Experiments under four temperature conditions are carried out and the corresponding parameter values are shown in **Table 2**.

Temperatures, °C Parameters	$T_0 = 23$	$T_0 = 23$	$T_0 = 27$	$T_0 = 27$
	$T_1 = 81$	$T_1 = 111$	$T_1 = 147$	$T_1 = 177$
r_{cc}/mOhm	0.6	0.7	0.7	0.8
r_{ch}/mOhm	0.6	0.7	0.7	0.8
$R_{th,c}/\text{K}\cdot\text{W}^{-1}$	31.0	31.5	30.9	31.0
$R_{th,h}/\text{K}\cdot\text{W}^{-1}$	23.5	23.3	22.8	23.0

Table 2. Thermal resistances and contact electrical resistances under different temperatures.

3.2. Test setup

System for measuring output performance of thermoelectric modules was established, mainly including electric heating plate controlled by PID, adjustable load, circulatory cooling unit, thermal imaging device, temperature and voltage data acquisition units, etc., with its basic structure as shown in **Figure 2**. Electric heating plate is used as heat source, with temperature control precision of ± 0.1 K and temperature ranging from room temperature to 773 K. Cooling unit, which consists of heat sink, water tank, flow meter and flow valve, etc., takes cold water as the coolant. Heat sink is made of red-copper and its temperature could be adjusted by controlling flowrate of cooling water. In addition, some thermal conductive filler is pasted on both sides of module to reduce thermal resistance between module, heat source and heat sink. Electrical current in the circuit is obtained via measuring voltage on both ends of sampling resistor (metal film precision resistor: 0.2 Ohm, precision of $\pm 1\%$).

Voltage and temperature signal are acquired by 9207 and 9214 acquisition card of National Instruments (NI) Company, with precision of $\pm 0.5\%$. Data to be acquired include as follows: (1) temperature of heat source and heat sink; (2) temperature of the coolant (water) inside heat sink and water tank; (3) voltage on adjustable load and sampling resistor. K-type thermocou-

the real values. Otherwise, when ΔT is large, material physical parameters change within a large scale, leading to a great deviation of calculations.

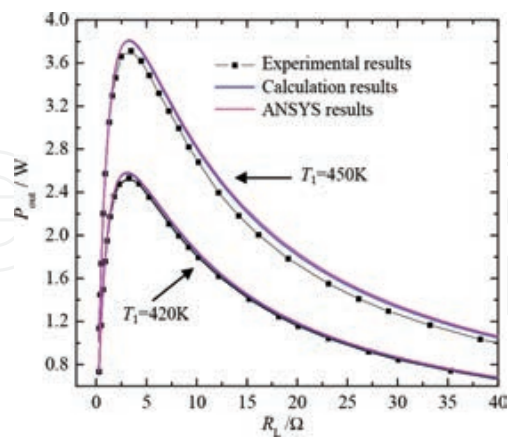


Figure 3. Dependences of output power on load resistance: calculations, experiments and ANSYS at $T_0 = 300$ K.

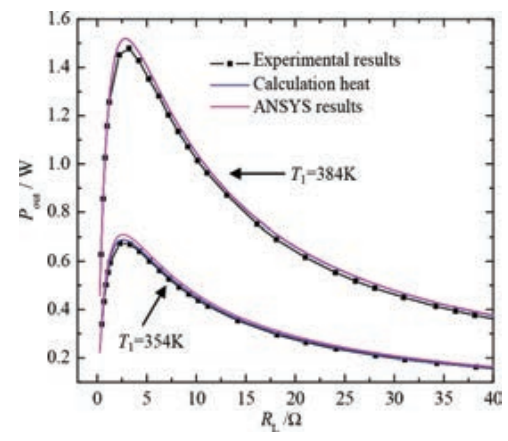


Figure 4. Dependences of output power on load resistance: calculations, experiments and ANSYS at $T_0 = 296$ K.

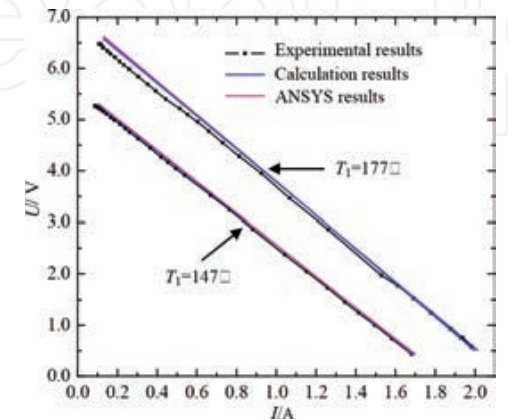


Figure 5. I-V characteristics of thermoelectric module: calculations, experiments and ANSYS at $T_0 = 300$ K.

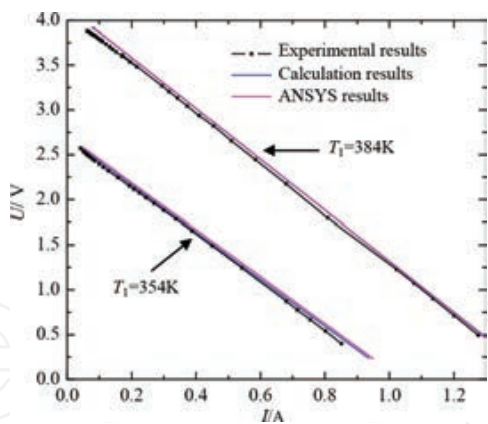


Figure 6. I-V characteristics of thermoelectric module: calculations, experiments and ANSYS at $T_0 = 296\text{ K}$.

4. Introduction to TEG simulation in ANSYS

4.1. TEG cell model

By software simulation, TEG performance can be achieved both in thermal and in electrical aspects. But it is not direct to cognize and understand the influence of thermoelectric effects, when compared with the above physical model. In this part, TEG cell model is set up by ANSYS, and geometry and meshing methods are illustrated in **Figure 7**. Thickness and cross-sectional area of thermoelements are 1.6 mm and 1.4 mm × 1.4 mm, respectively. Other geometry parameters are shown in **Figure 7**. Thermoelectric module consists mainly of p-n thermoelements, current-conducting copper straps and ceramic substrates for heat conducting and electric insulation. Thermoelements and copper strap are meshed by element SOLID226 in ANSYS. This type of element contains 20 nodes with voltage and temperature as the degrees of freedom. It can simulate 3D thermal-electrical coupling field. Element SOLID90 is used to mesh ceramic substrate. It has 20 nodes with temperature as the degree of freedom. Load resistance is simulated by element CIRCU124.

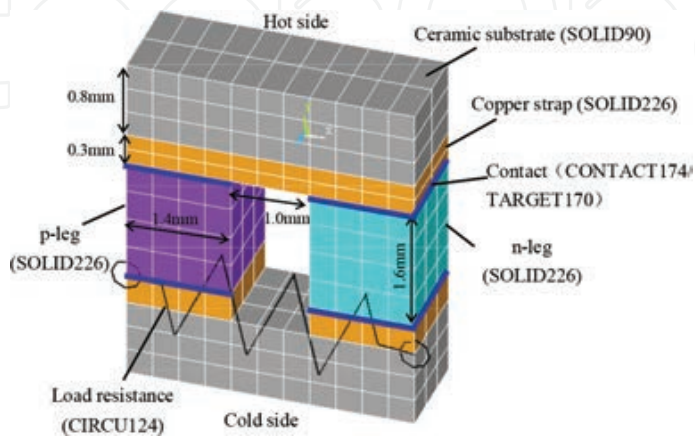


Figure 7. The geometry of TEG cell in ANSYS and its mesh.

Contact properties of the leg-strap junction are implemented with element pairs CONTACT174/TARGET170. Detailed finite element formulations in ANSYS are introduced in [4], and the range of contact thermal conductivity and electrical resistivity is explicated in [11].

4.2. APDL codes for TEG simulation

ANSYS Parametric Design Language (APDL) is widely used for programed simulation. The following APDL codes have taken temperature variation of materials properties, thermal contact and thermal radiation (although its influence is very weak) into consideration. According to the practical requirements, the readers could use the code more concisely by neglecting certain physical effects. The unit referring to length is meter and the temperature unit is Celsius.

! defining the TEG cell dimensions

ln=1.6e-3 ! n-type thermoelement thickness

lp=1.6e-3 ! p-type thermoelement thickness

wn=1.4e-3 ! p-type thermoelement width

wp=1.4e-3 ! p-type thermoelement width

d=1.0e-3 ! Distance between the thermoelements

hs=0.2e-3 ! copper strap thickness

hc=1e-3 ! substrate thickness

! definition of several physical parameters

rsvx=1.8e-8 ! copper electrical resistivity

kx=200 ! copper thermal conductivity

kxs=24 ! substrate thermal conductivity

T1=250 ! temperature of heat source

T0=30 ! temperature of heat sink

Toffst=273 ! temperature offset

! defining TEG output parameters and the load

**dim,P0,array,1 ! defining P0 as the output power*

**dim,R0,array,1 ! defining R0 as the load*

**dim,Qh,array,1 ! defining Qh as the heat flow into the TEG cell*

**dim,I,array,1 ! defining I as the current*

**dim,enta,array,1 ! defining enta as the energy efficiency*

```

*ofill,R0(1),ramp,0.025 ! setting the load (Ohm)

! pre-processing before calculation, defining element type, building the structure and meshing
/PREP7

toffst,Toffst ! set temperature offset

et,1,226,110 ! 20-node thermoelectric brick element
et,2,shell57 ! shell57 element for radiation simulation
et,3,conta174 ! conta174 element for contact simulation
et,4,targe170 ! target170 element for contact simulation

keyopt,3,1,4 ! taking temperature and voltage as the degree of freedom
keyopt,3,9,0
keyopt,3,10,1
keyopt,4,2,0
keyopt,4,3,0

! Temperature data points
mptemp,1,25,50,75,100,125,150
mptemp,7,175,200,225,250,275,300
mptemp,13,325,350

! Seebeck coefficient of the n-type material ( $V \cdot K^{-1}$ )
mpdata,sbkx,1,1,-160e-6,-168e-6,-174e-6,-180e-6,-184e-6,-187e-6
mpdata,sbkx,1,7,-189e-6,-190e-6,-189e-6,-186.5e-6,-183e-6,-177e-6
mpdata,sbkx,1,13,-169e-6,-160e-6

! electrical resistivity of the n-type material ( $\Omega \cdot m$ )
mpdata,rsvx,1,1,1.03e-5,1.06e-5,1.1e-5,1.15e-5,1.2e-5,1.28e-5
mpdata,rsvx,1,7,1.37e-5,1.49e-5,1.59e-5,1.67e-5,1.74e-5,1.78e-5
mpdata,rsvx,1,13,1.8e-5,1.78e-5

! thermal conductivity of the n-type material ( $m \cdot K^{-1}$ )
mpdata,kxx,1,1,1.183,1.22,1.245,1.265,1.265,1.25
mpdata,kxx,1,7,1.22,1.19,1.16,1.14,1.115,1.09
mpdata,kxx,1,13,1.06,1.03

! Seebeck coefficient of the p-type material ( $V \cdot K^{-1}$ )

```


mpdata,sbkx,2,1,200e-6,202e-6,208e-6,214e-6,220e-6,223e-6

mpdata,sbkx,2,7,218e-6,200e-6,180e-6,156e-6,140e-6,120e-6

mpdata,sbkx,2,13,101e-6,90e-6

! electrical resistivity of the p-type material (Ohm*m)

mpdata,rsvx,2,1,1.0e-5,1.08e-5,1.18e-5,1.35e-5,1.51e-5,1.7e-5

mpdata,rsvx,2,7,1.85e-5,1.98e-5,2.07e-5,2.143e-5,2.15e-5,2.1e-5

mpdata,rsvx,2,13,2.05e-5,2.0e-5

! thermal conductivity of the p-type material (m* K⁻¹)

mpdata,kxx,2,1,1.08,1.135,1.2,1.25,1.257,1.22

mpdata,kxx,2,7,1.116,1.135,1.13,1.09,1.12,1.25

mpdata,kxx,2,13,1.5,2.025

! material property for cooper strap

mp,rsvx,3,rsvx

mp,kxx,3,kx

! material property for the substrate

mp,kxx,4,kxs

! radiation property for the p-n materials

mp,emis,5

! contact friction coefficient

mp,mu,6,0

! build the TEG cell structure

block,d/2,wn+d/2,-ln,0,,t

block,-(wp+d/2),-d/2,-lp,0,,t

block,d/2,wn+d/2,,hs,,t

block,-(wp+d/2),-d/2,,hs,,t

block,-d/2,d/2,,hs,,t

block,-(wp+d/2),-d/2,-lp,-(lp+hs),,t

block,d/2,wn+d/2,-ln,-(ln+hs),,t

block,-(wp+d/2),wn+d/2,hs,hs+hc,,t

block,-(wp+d/2),wn+d/2,-(lp+hs),-(lp+hs+hc),,t

! glue the copper strap and the substrate

vsel,s,loc,y,0,hs

vsel,a,loc,y,hs,hs+hs

vglue,all

allsel

vsel,s,loc,y,-lp-hs,-lp

vsel,a,loc,y,-lp-hs-hc,-lp-hs

vglue,all

allsel

! meshing the TEG cell structure

numcmp,all

mshape,0,3d

mshkey,1

type,1

mat,3

lsl,s,loc,x,-d/2,d/2

lsl,r,loc,y,0

lsl,r,loc,z,t

lesize,all,d/3

vsel,s,loc,x,-d/2,d/2

vsel,r,loc,y,0,hs

vsweep,all

allsel

esize,ww/3

type,1

mat,3

vsel,s,loc,y,0,hs

vsel,u,loc,x,-d/2,d/2

vsweep,all

vsel,s,loc,y,-lp-hs,-lp

```
vsweep,all
type,1
mat,1
vsel,s,loc,x,d/2,d/2+wn
vsel,r,loc,y,-ln,0
vmesh,all
mat,2
vsel,s,loc,x,-(wp+d/2),-d/2
vsel,r,loc,y,-lp,0
vmesh,all
type,1
mat,4
vsel,s,loc,y,hs,hs+hc
vsel,a,loc,y,-lp-hs-hc,-lp-hs
vsweep,all
allsel
! defining the contact parameters
r,5 ! selecting the thermal contact conductivity and resistivity
RMORE,
rmore,,7e5 ! setting the thermal contact conductivity
rmore,0.67e8,0.5 ! setting the thermal contact resistivity
! defining the contact layer between p-leg and upper copper strap
vsel,s,loc,y,0,hs
asel,s,ext
asel,r,loc,y,0
nsla,s,1
nsl,r,loc,x,-(wp+d/2),-d/2
type,3
mat,6
real,5
```

esurf

allsel

! defining the target layer between p-leg and upper copper strap

vsel,s,mat,,2

asel,s,ext

asel,r,loc,y,0

nsla,s,1

type,4

mat,6

esurf

allsel

! defining the contact layer between n-leg and upper copper strap

vsel,s,loc,y,0,hs

asel,s,ext

asel,r,loc,y,0

nsla,s,1

nsl,r,loc,x,d/2,d/2+wn

type,3

mat,6

real,5

esurf

allsel

! defining the target layer between n-leg and upper copper strap

vsel,s,mat,,1

asel,s,ext

asel,r,loc,y,0

nsla,s,1

type,4

mat,6

esurf

```
allsel
! defining the contact layer between p-leg and bottom copper strap
vsel,s,loc,y,-hs-lp,-lp
vsel,r,loc,x,-wp-d/2,-d/2
asel,s,ext
asel,r,loc,y,-lp
nsla,s,1
type,3
mat,6
real,5
esurf
allsel
! defining the target layer between p-leg and bottom copper strap
vsel,s,mat,,2
asel,s,ext
asel,r,loc,y,-lp
nsla,s,1
type,4
mat,6
esurf
allsel
! defining the contact layer between n-leg and bottom copper strap
vsel,s,loc,y,-hs-ln,-ln
vsel,r,loc,x,d/2,d/2+wn
asel,s,ext
asel,r,loc,y,-ln
nsla,s,1
type,3
mat,6
real,5
```

```
esurf
allsel
! defining the target layer between n-leg and bottom copper strap
vsel,s,mat,,1
asel,s,ext
asel,r,loc,y,-ln
nsla,s,1
type,4
mat,6
esurf
allsel
! defining the shell element for radiation simulation, outputting radiation matrix
! defining the shell element for copper strap
type,2
aatt,3,,2
asel,s,loc,x,-(wp+d/2),wn+d/2
asel,r,loc,y,0,hs
asel,u,loc,y,0
asel,u,loc,y,hs
amesh,all
allsel
asel,s,loc,x,-d/2,d/2
asel,r,loc,y,0
amesh,all
allsel
aatt,3,,2
asel,s,loc,x,-(wp+d/2),wn+d/2
asel,r,loc,y,-lp-hs,-lp
asel,u,loc,y,-lp
asel,u,loc,y,-lp-hs
```

```
amesh,all
allsel
aatt,4,,2
asel,s,loc,x,-d/2,d/2
asel,r,loc,y,-lp-hs
amesh,all
! defining the shell element for p-n thermoelements
allsel
aatt,5,,2
asel,s,loc,x,-(wp+d/2),wn+d/2
asel,r,loc,y,-lp,0
asel,u,loc,y,-lp
asel,u,loc,y,0
amesh,all
! defining the space node for radiation simulation
n,10000,0,0,3e-3
fini
! using radiation matrix method
/aux12
emis,3,1 ! setting the emissivity
emis,4,1
emis,5,1
allsel
geom,0
stef,5.68e-8 ! setting the Stefan-Boltzmann constant
vtype,hidden
space,10000
write,teg,sub ! outputting the radiation super element
fini
/prep7
```

```

! deleting the shell elements and the corresponding mesh
allsel
asel,s,type,,2
aclear,al
etdele,2
allsel
et,5,matrix50,1 ! defining radiation matrix element
! defining boundary conditions and the load
nsel,s,loc,y,hs+hc ! TEG cell hot side
cp,1,temp,all ! coupling of temperature degree of freedom
nh=ndnext(0) ! getting the master node
d,nh,temp,Th ! setting the temperature constraint to the hot side
nsel,all
nsel,s,loc,y,-(ln+hs+hc) ! selecting the TEG cell cold side
d,all,temp,Tc ! setting the temperature constraint to the cold side
nsel,s,loc,y,-(ln+hs),-ln
nsel,r,loc,x,d/2+wn
cp,3,volt,all ! electrical coupling
nn=ndnext(0) ! getting the master node
d,nn,volt,0 ! setting the ground connection node
nsel,all
nsel,s,loc,y,-(lp+hs),-lp
nsel,r,loc,x,-(wp+d/2)
cp,4,volt,all ! ! electrical coupling
np=ndnext(0) ! getting the master node
nsel,all
type,5
allsel
d,10000,temp,300 ! setting the temperature of the space node
se,teg,sub ! reading the radiation super element

```



```
et,6,CIRCU124,0 ! setting the load resistor element
fini
/prep7
! setting the load value and property
r,1,R0(1)
type,6
real,1
numcmp,all
e,np,nn
esel,s,type,,6
circu_num=elnex(0) !getting circuit element number
allsel
fini
! starting the calculation
/SOLU
antype,static ! solution type
cnvtol,heat,1,1.e-3 ! setting the converging value for heat condition
cnvtol,amps,1,1.e-3 ! setting the converging value for the current
neqit,50 ! calculation iteration step
solve ! starting solving
fini
*get,P0(1),elem,circu_num,nmisc,1 ! getting the output power of the TEG cell
*get,Qh(1),node,nh,rf,heat ! getting the heat flow into the TEG cell
*get,I(1),elem,circu_num,smisc,2 ! getting the current
*voper,enta,P0,div,Qh ! calculating the energy efficiency of the TEG cell
```

5. Conclusions

The built one-dimensional model, which is validated by test results, can calculate TEG output power and energy efficiency accurately. By simplifying this model, it is convenient to analyze

influences of different thermal and electrical parameters on TEG performance. And basic factors to enhance TEG output power and energy efficiency are extracted. At last, ANSYS simulation considering thermal contact and radiation effects for TEGs is introduced briefly, and basic APDL codes are shared.

Author details

Fuqiang Cheng

Address all correspondence to: chengfq101@aliyun.com

1 Key Laboratory of Fault Diagnosis and Maintenance of In-orbit Spacecraft, Xi'an Satellite Control Center, Xi'an, China

2 Equipment Academy, Beijing, China

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